- 1. A beam with an equilateral cross section is loaded by a statically equivalent load of P acting at the centroid and directed along the *x*-axis as shown in Fig. 1. By symmetry,  $x_{\rm cf} = 0$ . Show that  $y_{\rm cf} = 0$ , i.e., the shear center is at the centroid, by using the following procedure:
  - (a) State the value of  $I_{xy}$  (without proof), and use it to find the 'curvatures'

$$\kappa_x = \frac{I_{xx}W_x + I_{xy}W_y}{E(I_{xx}I_{yy} - I_{xy}^2)}; \quad \kappa_y = \frac{I_{yy}W_y + I_{xy}W_x}{E(I_{xx}I_{yy} - I_{xy}^2)},$$

in terms of  $I_{xx}$  and  $I_{yy}$  (which you need not evaluate).

(b) Find the functions g(x) and f(y) such that the boundary conditions

$$\left[\frac{1}{2}E\kappa_x x^2 - f(y)\right]\frac{dy}{ds} = 0; \quad \left[\frac{1}{2}E\kappa_y y^2 + g(x)\right]\frac{dx}{ds} = 0,$$

are satisfied. (Hint: See if f(y) can be determined by considering part of the surface).

(c) Given that  $\nu = 0.5$ , use the governing differential equation

$$\boldsymbol{\nabla}^2 \phi = -2G\nu\kappa_y x - \frac{\partial g}{\partial x} + 2G\nu\kappa_x y - \frac{\partial f}{\partial y},$$

to find  $\phi$  (Hint:  $\phi = 0$  on the boundary). Substitute this expression for  $\phi$  in the equation

$$y_{\rm cf} = -\int_A \left[ \frac{2\phi}{W_x} - \frac{I_{xx}}{I_{xx}I_{yy} - I_{xy}^2} x^2 y + \frac{I_{xy}}{I_{xx}I_{yy} - I_{xy}^2} x y^2 \right] \, dA,$$

to find the location of the shear center. If the load P passes through the centroid, then the same  $\phi$  that is determined above can be used to find the stresses.

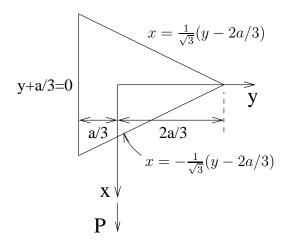


Figure 1: Problem 1.

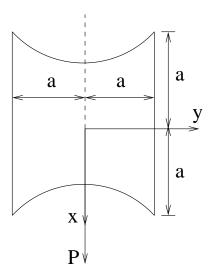


Figure 2: Problem 2.

- 2. A beam of the doubly symmetric cross section comprised of two vertical sides  $y = \pm a$ , and two hyperbolas  $(1 + \nu)x^2 \nu y^2 = a^2$  is loaded by a statically equivalent load of P acting at the centroid and directed along the x-axis as shown in Fig. 2.
  - (a) State with justification (but without mathematical proof), the value of twist at the centroid  $\alpha$ .
  - (b) State the value of  $I_{xy}$  (without proof), and use it to find the 'curvatures'

$$\kappa_x = \frac{I_{xx}W_x + I_{xy}W_y}{E(I_{xx}I_{yy} - I_{xy}^2)}; \quad \kappa_y = \frac{I_{yy}W_y + I_{xy}W_x}{E(I_{xx}I_{yy} - I_{xy}^2)},$$

in terms of  $I_{xx}$  and  $I_{yy}$  (which you need not evaluate).

(c) Find the functions g(x) and f(y) such that the boundary conditions

$$\left[\frac{1}{2}E\kappa_x x^2 - f(y)\right]\frac{dy}{ds} = 0; \quad \left[\frac{1}{2}E\kappa_y y^2 + g(x)\right]\frac{dx}{ds} = 0,$$

are satisfied.

(d) Use the governing differential equation

$$\nabla^2 \phi = -2G\nu\kappa_y x - \frac{\partial g}{\partial x} + 2G\nu\kappa_x y - \frac{\partial f}{\partial y}$$

and the appropriate boundary condition to find  $\phi$  (Hint: Take  $\phi = a_1 + a_2x + a_3y$ , and find  $a_1$ ,  $a_2$  and  $a_3$ .)

(e) Find the stresses using the relations

$$\tau_{xz} = \frac{\partial \phi}{\partial y} + f(y) - \frac{1}{2}E\kappa_x x^2,$$
  
$$\tau_{yz} = -\frac{\partial \phi}{\partial x} - g(x) - \frac{1}{2}E\kappa_y y^2,$$

and find  $(\tau_{xz})_{\text{max}}$ , and the location where it occurs.