

Indian Institute of Science

ME 242: Final Exam

Date: 22/1/21.

Duration: 9.15 a.m.–1.15 p.m.

Maximum Marks: 100

1. Let $\mathbf{W} \in \text{Skw}$ and $\mathbf{Q} \in \text{Orth}^+$. (15)

- Determine whether $\mathbf{Q}\mathbf{W}^2\mathbf{Q}^T$ is symmetric, skew-symmetric, orthogonal or none of the above.
- Determine the eigenvalue/eigenvectors of $\mathbf{Q}\mathbf{W}^2\mathbf{Q}^T$. If the eigenvectors are nonunique, *choose* a set of eigenvectors, and present your choice.
- It is possible for tensors to fall into two categories. For example, the diagonal matrix $\mathbf{diag}[1, -1, -1]$ is both symmetric and orthogonal. Determine if $\mathbf{Q}\mathbf{W}^2\mathbf{Q}^T$ can be orthogonal. If it can be, give an example, if it cannot be, present a mathematical justification.

2. A circular hole of radius a in an unbounded domain is subjected to a traction distribution $\mathbf{t} = (p_0 + p_1 \cos \theta)\mathbf{e}_r$ (with (r, θ) denoting polar coordinates) on its surface. The Airy stress function and the corresponding stresses are given by (15)

$$\begin{aligned}\phi &= F_2\theta + \frac{A_2a^3(\nu - 1)\cos\theta}{2r} + \frac{B_2a^3(1 - \nu)\sin\theta}{2r} + C_2r^3\cos\theta + a^2F_1\log r \\ &\quad + r\cos\theta[2H_2\theta + J_2a(\nu - 1)\log r] + r\sin\theta[2J_2a\theta + aH_2(1 - \nu)\log r], \\ \tau_{rr} &= \frac{F_1a^2}{r^2} + 2C_2r\cos\theta + \frac{A_2a^3(1 - \nu)\cos\theta}{r^3} + \frac{J_2a(1 + \nu)\cos\theta}{r} \\ &\quad - \frac{B_2a^3(1 - \nu)\sin\theta}{r^3} - \frac{H_2(1 + \nu)a\sin\theta}{r}, \\ \tau_{\theta\theta} &= -\frac{F_1a^2}{r^2} + 6C_2r\cos\theta - \frac{A_2a^3(1 - \nu)\cos\theta}{r^3} + \frac{J_2a(\nu - 1)\cos\theta}{r} \\ &\quad + \frac{B_2a^3(1 - \nu)\sin\theta}{r^3} + \frac{H_2(1 - \nu)a\sin\theta}{r}, \\ \tau_{r\theta} &= \frac{F_2}{r^2} + 2C_2r\sin\theta + \frac{A_2a^3(1 - \nu)\sin\theta}{r^3} - \frac{J_2a(1 - \nu)\sin\theta}{r} \\ &\quad + \frac{B_2a^3(1 - \nu)\cos\theta}{r^3} - \frac{H_2a(1 - \nu)\cos\theta}{r}.\end{aligned}$$

Determine the various constants (total seven in number) in the above formulation.

3. Consider the torsion of a bar whose cross section is a triangle with angles (30)

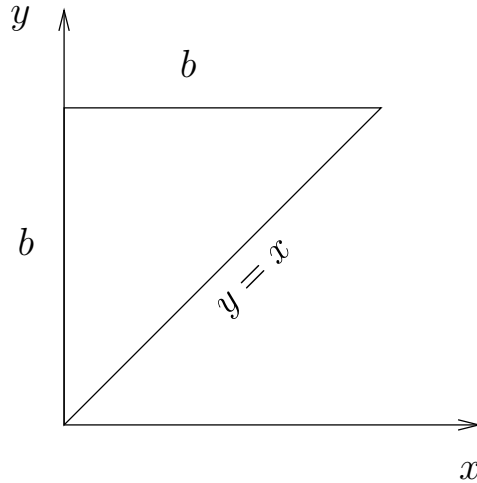


Figure 1: Torsion of a bar whose cross-section is a triangle with angles 45°-45°-90°.

45°-45°-90° as shown in Fig. 1. The Prandtl stress function is given by

$$\frac{\phi}{G\alpha} = c_0x(y-x) - \sum_{m=1}^{\infty} A_m [\sin(\beta_m x) \sinh(\beta_m y) - \sin(\beta_m y) \sinh(\beta_m x)],$$

- (a) Determine the constants c_0 , A_m and β_m (where the constants with subscript m are dependent on m). Some of these constants can be in the form of an integral with appropriate limits; you need not evaluate these integrals.
 - (b) Write the expression for the torsional rigidity as an integral with appropriate limits. *Do not* evaluate this integral.
4. A cracked beam of circular cross section of radius b with the crack extending from the center to the periphery, and along the entire length $[0, L]$ is loaded by a statically equivalent load of P acting at the centroid and directed along the y -axis as shown in Fig. 2. (40)

- (a) State with justification (but without mathematical proof), the value of twist at the centroid α .
- (b) State the value of I_{xy} (without proof), and use it to find the ‘curvatures’

$$\kappa_x = \frac{I_{xx}W_x + I_{xy}W_y}{E(I_{xx}I_{yy} - I_{xy}^2)}; \quad \kappa_y = \frac{I_{yy}W_y + I_{xy}W_x}{E(I_{xx}I_{yy} - I_{xy}^2)},$$

in terms of I_{xx} and I_{yy} (which you need not evaluate).

- (c) Use the formulae

$$\left[\frac{1}{2}E\kappa_x x^2 - f(y) \right] \frac{dy}{ds} - \left[\frac{1}{2}E\kappa_y y^2 + g(x) \right] \frac{dx}{ds} = 0,$$

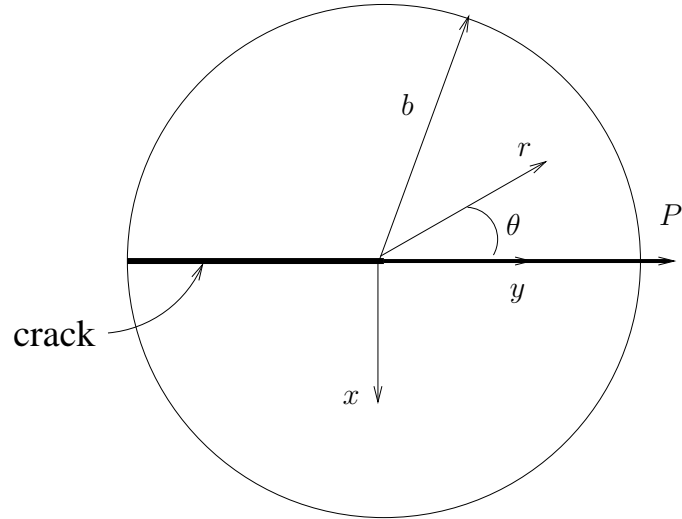


Figure 2: Problem 4.

$$\nabla^2 \phi = -2G\nu\kappa_y x - \frac{dg}{dx} + 2G\nu\kappa_x y - \frac{df}{dy} - 2G\alpha,$$

to determine ϕ .

- (d) Find the relations for τ_{rz} and $\tau_{\theta z}$ in terms of τ_{xz} , τ_{yz} and (r, θ) (*do not* substitute for τ_{xz} and τ_{yz} in these expressions.)

Some relevant formulae

$$w_i = -\frac{1}{2}\epsilon_{ijk}W_{jk},$$

$$W_{ij} = -\epsilon_{ijk}w_k,$$

$$\mathbf{W} = |\mathbf{w}|(\mathbf{r} \otimes \mathbf{q} - \mathbf{q} \otimes \mathbf{r}), \quad \{\mathbf{w}/|\mathbf{w}|, \mathbf{q}, \mathbf{r}\} \text{ orthonormal}$$