

Indian Institute of Science

ME 242: Midsemester Test

Date: 24/9/05.

Duration: 9.30 a.m.–11.00 p.m.

Maximum Marks: 100

Instructions:

1. Justify all your steps.
2. The points for each question are indicated in the right margin.
1. Show using indicial notation that $\text{tr}(\mathbf{RS}) = \text{tr}(\mathbf{SR})$ and $\det(\mathbf{RS}) = (\det \mathbf{R})(\det \mathbf{S})$. Use (25) these results to show that the eigenvalues of \mathbf{RS} and \mathbf{SR} are the same.
2. Let the velocity field be given by $\mathbf{v} = (\mathbf{x}/|\mathbf{x}|)$, where \mathbf{x} is the spatial coordinate. Find the (25) acceleration \mathbf{a} using the Eulerian framework.
3. The goal of this problem is to derive an “energy” equation over the domain V from the (30) governing equations.
 - (a) If $\boldsymbol{\tau}$ is the stress tensor and \mathbf{u} is the displacement vector, both defined over V , derive a relation between $\nabla \cdot (\boldsymbol{\tau}^t \mathbf{u})$, $\boldsymbol{\tau} : \nabla \mathbf{u}$ and $\mathbf{u} \cdot (\nabla \cdot \boldsymbol{\tau})$ (where the gradients are with respect to \mathbf{x}).
 - (b) Using the symmetry of $\boldsymbol{\tau}$, find a relation between $\boldsymbol{\tau} : \nabla \mathbf{u}$ and $\boldsymbol{\tau} : \boldsymbol{\epsilon}(\mathbf{u})$, where $\boldsymbol{\epsilon}(\mathbf{u}) := (\nabla \mathbf{u} + (\nabla \mathbf{u})^t)/2$, and substitute it into the relation derived in (a).
 - (c) Substitute the final relation obtained in (b) into the relation

$$\int_V \mathbf{u} \cdot (\nabla \cdot \boldsymbol{\tau} + \rho \mathbf{b}) dV = 0,$$

and use the definition of transpose, the divergence theorem and the Cauchy relation to find a relation between the “strain energy” $\frac{1}{2} \int_V \boldsymbol{\tau} : \boldsymbol{\epsilon}(\mathbf{u}) dV$, and the “work done by the forces” given by $\int_S \mathbf{t} \cdot \mathbf{u} dS + \int_V \rho \mathbf{b} \cdot \mathbf{u} dV$, where \mathbf{t} is the traction vector.

4. A hollow cylinder with inner radius r_1 and outer radius r_2 under “plane strain” conditions (20) (i.e., $u_z = 0$) is subjected to a uniform pressure p at the inner boundary, as shown in Fig. 1. The displacement solution is given by

$$u_r = c_1 r + \frac{c_2}{r}, \quad u_\theta = 0, \quad u_z = 0,$$

where c_1 and c_2 are constants to be determined (in part (c) below).

- (a) Find the strains using the relations

$$\epsilon_{rr} = \frac{\partial u_r}{\partial r}; \quad \epsilon_{zz} = \frac{\partial u_z}{\partial z}; \quad \epsilon_{\theta\theta} = \frac{u_r}{r},$$

with the other strain components zero.

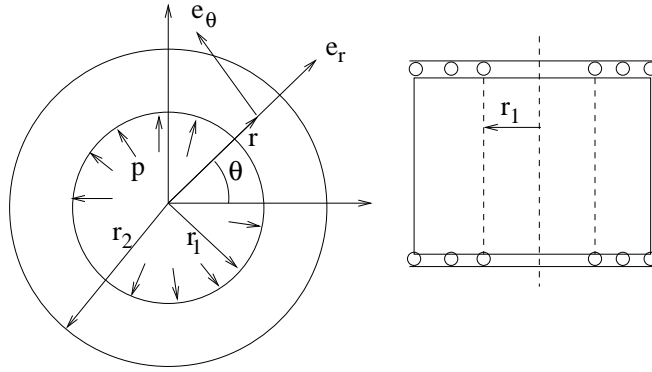


Figure 1: Cylinder subjected to internal pressure p under plane strain conditions.

- (b) Using the constitutive relation $\boldsymbol{\tau} = \lambda(\text{tr } \boldsymbol{\epsilon})\mathbf{I} + 2\mu\boldsymbol{\epsilon}$, find expressions for the components of $\boldsymbol{\tau}$.
- (c) Write the traction boundary conditions (referred to the \mathbf{e}_r - \mathbf{e}_θ - \mathbf{e}_z system) on the inner and outer surface of the cylinder, i.e., at $r = r_1$ and at $r = r_2$, and on the top surface.
- (d) Use these traction boundary conditions to determine the stress field.
- (e) Find the *total* axial force F_z exerted on the top wall by the cylinder.

Some relevant formulae

$$\det \mathbf{T} = \epsilon_{ijk} T_{i1} T_{j2} T_{k3} = \epsilon_{ijk} T_{1i} T_{2j} T_{3k},$$

$$\epsilon_{pqr}(\det \mathbf{T}) = \epsilon_{ijk} T_{ip} T_{jq} T_{kr} = \epsilon_{ijk} T_{pi} T_{qj} T_{rk}.$$