## Indian Institute of Science, Bangalore

## ME 243: Endsemester Exam

Date: 12/12/12. Duration: 9.30 a.m.–12.30 p.m. Maximum Marks: 100

## Instructions:

You may directly use the formulae at the back.

- 1. We saw in the test that  $\{I, W, W^2\}$  is linearly independent. Is  $\{I, Q, Q^T\}$  (15) linearly dependent or linearly independent? If not linearly independent, under what conditions is it linearly dependent?
- 2. Assuming the relation  $\mathbf{x}^* = \mathbf{Q}\mathbf{x} + \mathbf{c}$  between the position vectors  $\mathbf{x}$  and (35)  $\mathbf{x}^*$  in two frames of reference, find the relation between  $\nabla_{\mathbf{x}^*} \cdot \mathbf{v}^*$  and  $\nabla_{\mathbf{x}} \cdot \mathbf{v}$ . Starting from the *integral form* of the balance of angular momentum, *derive* the consequences of assuming 'frame indifference' of this integral form (balance law has the same form in all frames of reference). Do not assume balance of mass or linear momentum to hold. You may assume that  $dV^* = dV$ , and

$$egin{aligned} & 
ho^*(oldsymbol{x}^*,t) = 
ho(oldsymbol{x},t), \quad oldsymbol{ au}^T = oldsymbol{ au}, \quad oldsymbol{t} = oldsymbol{ au}oldsymbol{n}, \\ & oldsymbol{t}^*(oldsymbol{x}^*,t,oldsymbol{n}^*) = oldsymbol{Q}oldsymbol{t}(oldsymbol{x},t,oldsymbol{n}), \quad oldsymbol{b}(oldsymbol{b}^* - oldsymbol{a}^*) = oldsymbol{Q}(oldsymbol{b} - oldsymbol{a}), \\ & oldsymbol{ au}^* = oldsymbol{Q}oldsymbol{ au}oldsymbol{R}, \quad oldsymbol{
au}^T = oldsymbol{ au}oldsymbol{Q}(oldsymbol{b} - oldsymbol{a}), \\ & oldsymbol{ au}^* = oldsymbol{Q}oldsymbol{ au}oldsymbol{R}, \quad oldsymbol{
au}^T = oldsymbol{ au}oldsymbol{
au}oldsymbol{
au}oldsymbol{n} = oldsymbol{A}oldsymbol{
au}oldsymbol{b}(oldsymbol{b} - oldsymbol{a}), \\ & oldsymbol{ au}^* = oldsymbol{ au}oldsymbol{
au}oldsymbol{R}, \quad oldsymbol{
au}oldsymbol{
au}^T = oldsymbol{ au}oldsymbol{
au}oldsymbol{
au}oldsymbol{n} = oldsymbol{A}oldsymbol{
au}oldsymbol{
au}oldsymbol{n} = oldsymbol{
au}oldsymbol{
au}oldsymbol{$$

(Hint: Careful with the use of transport theorems.)

- 3. Let x<sup>\*</sup>-y<sup>\*</sup> be a 'fixed' frame of reference with body force b = ge<sub>x</sub><sup>\*</sup>. A rod of (25) length l and mass per unit length m, which is pinned at one end, oscillates in the x<sup>\*</sup>-y<sup>\*</sup> plane as shown in Fig. 1. The pin support itself oscillates along the y<sup>\*</sup> axis with its distance from the x<sup>\*</sup>-y<sup>\*</sup> origin being given by u = u<sub>0</sub> sin ωt. Using the appropriate transformation laws, find the body force field in the x-y frame. Approximating the rod as a one-dimensional rigid body (ρdV ≡ m dx), and applying the appropriate balance laws in integral form in the x-y frame, find the governing equation for the angle θ, and the reactions exerted at the pinned-support with respect to the x-y axes. Linearize the governing equation for θ and solve it.
- 4. We have seen that the equation of rigid motion in a Cartesian system is (25) given by  $\boldsymbol{v} = \boldsymbol{W}\boldsymbol{x} + \boldsymbol{c}$ , where  $\boldsymbol{W} \in \text{Skw}$ . By using appropriate tensorial transformations, find the expressions for  $(v_r, v_\theta, v_z)$  for a rigid motion. Your final expressions for these components should be functions of  $(r, \theta, z)$  (including some constants). Now, solve the following problem. A Newtonian, incompressible viscous fluid is contained in a rotating cylindrical container of radius *a* rotating with constant angular speed  $\omega_0$  as shown in Fig. 2. The



Figure 1: Oscillating rod.



Figure 2: Problem 4.

fluid inside reaches to a steady-state velocity. Assuming  $v_z = 0$ , and assuming the velocity field in a *stationary* frame of reference with  $\mathbf{b} = \mathbf{0}$  to be of the form you have derived above, first find the velocity field and then the pressure field from the momentum equations assuming  $p|_{r=0} = 0$ .

## Some relevant formulae

$$\frac{d}{dt} \int_{V(t)} f(\boldsymbol{x}, t) \, dV = \int_{V(t)} \frac{\partial f}{\partial t} \, dV + \int_{S(t)} f(\boldsymbol{v} \cdot \boldsymbol{n}) \, dS,$$

$$\frac{d}{dt} \int_{V(t)} \rho f(\boldsymbol{x}, t) \, dV = \int_{V(t)} \rho \frac{Df}{Dt} \, dV = \int_{V(t)} \rho \left\{ \frac{\partial f}{\partial t} + \boldsymbol{v} \cdot (\boldsymbol{\nabla} f) \right\} \, dV,$$

$$\int_{V(t)} (\boldsymbol{x} \times \boldsymbol{W} \boldsymbol{x}) \phi \, dV = 0 \quad \forall \boldsymbol{W} \in \text{Skw } \& \; \forall V(t) \implies \phi = 0,$$

$$\boldsymbol{W} = |\boldsymbol{w}| \, (\boldsymbol{r} \otimes \boldsymbol{q} - \boldsymbol{q} \otimes \boldsymbol{r}),$$

$$\boldsymbol{Q} = \boldsymbol{I} + \sin \alpha \, \boldsymbol{W} + (1 - \cos \alpha) \boldsymbol{W}^{2},$$

$$\boldsymbol{b} = \boldsymbol{Q}^{T} \left[ \boldsymbol{b}^{*} - \boldsymbol{\ddot{c}} \right] - \boldsymbol{\dot{\Omega}} \times \boldsymbol{x} - \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \boldsymbol{x}) - 2\boldsymbol{\Omega} \times \boldsymbol{v}.$$

$$Q_{ij} = \boldsymbol{e}_{i}^{*} \cdot \boldsymbol{e}_{j},$$

$$\boldsymbol{\Omega} = \begin{bmatrix} \dot{\boldsymbol{e}}_{2} \cdot \boldsymbol{e}_{3} \\ \dot{\boldsymbol{e}}_{3} \cdot \boldsymbol{e}_{1} \\ \dot{\boldsymbol{e}}_{1} \cdot \boldsymbol{e}_{2} \end{bmatrix},$$

$$Q_{ij} = \bar{\boldsymbol{e}}_{i} \cdot \boldsymbol{e}_{j}.$$

$$D_{rr} = \frac{\partial u_{r}}{\partial r}, \qquad D_{r\theta} = \frac{1}{2} \begin{bmatrix} \frac{1}{r} \frac{\partial u_{r}}{\partial \theta} + r \frac{\partial}{\partial r} \left( \frac{u_{\theta}}{r} \right) \end{bmatrix},$$

$$D_{\theta\theta} = \frac{1}{r} \left( \frac{\partial u_{\theta}}{\partial \theta} + u_r \right), \qquad D_{\theta z} = \frac{1}{2} \left( \frac{\partial u_{\theta}}{\partial z} + \frac{1}{r} \frac{\partial u_z}{\partial \theta} \right),$$
$$D_{zz} = \frac{\partial u_z}{\partial z}, \qquad D_{rz} = \frac{1}{2} \left( \frac{\partial u_r}{\partial z} + \frac{\partial u_z}{\partial r} \right).$$

The momentum equations in r,  $\theta$  and z directions:

$$\frac{\partial v_r}{\partial t} + (\boldsymbol{v} \cdot \boldsymbol{\nabla})v_r - \frac{v_{\theta}^2}{r} = -\frac{1}{\rho}\frac{\partial p}{\partial r} + \nu \left[\boldsymbol{\nabla}^2 v_r - \frac{v_r}{r^2} - \frac{2}{r^2}\frac{\partial v_{\theta}}{\partial \theta}\right] + b_r,$$
  
$$\frac{\partial v_{\theta}}{\partial t} + (\boldsymbol{v} \cdot \boldsymbol{\nabla})v_{\theta} + \frac{v_r v_{\theta}}{r} = -\frac{1}{\rho r}\frac{\partial p}{\partial \theta} + \nu \left[\boldsymbol{\nabla}^2 v_{\theta} - \frac{v_{\theta}}{r^2} + \frac{2}{r^2}\frac{\partial v_r}{\partial \theta}\right] + b_{\theta},$$
  
$$\frac{\partial v_z}{\partial t} + (\boldsymbol{v} \cdot \boldsymbol{\nabla})v_z = -\frac{1}{\rho}\frac{\partial p}{\partial z} + \nu \boldsymbol{\nabla}^2 v_z + b_z.$$

where  $\nu = \mu/\rho$ , and

$$\boldsymbol{v} \cdot \boldsymbol{\nabla} \equiv v_r \frac{\partial}{\partial r} + \frac{v_\theta}{r} \frac{\partial}{\partial \theta} + v_z \frac{\partial}{\partial z}$$
$$\boldsymbol{\nabla}^2 \equiv \frac{1}{r} \frac{\partial}{\partial r} \left( r \frac{\partial}{\partial r} \right) + \frac{1}{r^2} \frac{\partial^2}{\partial \theta^2} + \frac{\partial^2}{\partial z^2}.$$