

# Indian Institute of Science, Bangalore

## ME 243: Endsemester Exam

**Date:** 29/1/21.

**Duration:** 9.15 a.m.–1.15 p.m.

**Maximum Marks:** 100

### Instructions:

You may directly use the formulae at the back.

1. We have seen that if  $\mathbf{Q} \in \text{Orth}^+$ , then  $\mathbf{Q}(\mathbf{u} \times \mathbf{v}) = (\mathbf{Q}\mathbf{u}) \times (\mathbf{Q}\mathbf{v})$ . Conversely, (20)  
if  $\mathbf{T}(\mathbf{u} \times \mathbf{v}) = (\mathbf{T}\mathbf{u}) \times (\mathbf{T}\mathbf{v}) \forall \mathbf{u}, \mathbf{v} \in \mathbb{R}^3$ , then is it necessarily true that  $\mathbf{T} \in \text{Orth}^+$ ? Consider *all* possibilities when you try to present your solution for  $\mathbf{T}$ .
2. In this problem, assume all field variables to be independent of  $Z$ . A cylinder (25)  
of inner radius  $a$ , outer radius  $b$ , and length  $L$  along the  $z$ -direction is fixed rigidly at the inner boundary  $r = a$ , and constrained by two rigid frictionless surfaces on the top and bottom surfaces  $z = 0$  and  $z = L$ . This cylinder is subjected to a moment  $M$  on the outer boundary  $r = b$  by the application of suitable tangential tractions which are independent of  $\theta$  and  $Z$  (see Fig. 1). Assume  $r = \alpha R$ ,  $z = Z$ , and assume a suitable mapping for  $\theta$  in terms of an unknown function. The material is incompressible, and the constitutive equation is given by  $\boldsymbol{\tau} = -p\mathbf{I} + \mu\mathbf{B}$ .
  - (a) Find the constant  $\alpha$  and the governing differential equation for your unknown function and any other unknown field. Do not attempt to solve any of these differential equations.
  - (b) State the boundary conditions for the unknown functions and the other unknown field.
3. The rigid rod shown in Fig. 2 will slide under the action of gravity assuming (30)  
that the floor and the wall are frictionless. A continuum enthusiast who wants the rod to remain stationary even under the action of gravity decides to accelerate the whole setup by a constant acceleration  $a$  along the  $x$ -direction as shown in Fig. 2.
  - (a) Determine if the rod can remain in static equilibrium *in the accelerating frame*  $x$ - $y$  (i.e.,  $\theta(t) = \theta_0$  where  $\theta_0$  is a constant), and if it can, is there a specific value of  $a$  for which it will remain in equilibrium? If there is, determine this value. For the purpose of carrying out the integrations, assume the rod to be one-dimensional, express the coordinates of a typical point  $(x, y)$  on the rod in terms the coordinate  $\xi$  which runs along the length as shown in the figure as  $x = \xi \cos \theta$ ,  $y = (L - \xi) \sin \theta$ , and assume  $\rho dV = m d\xi$ , where  $m$ , the mass per unit length, is a constant. *Do not* use the Euler equations. Solve using first principles.

- (b) For this part, assume that  $a \neq 0$  is such that the rod is *not* in equilibrium in the accelerating frame of reference (i.e.,  $\theta(t)$  varies with time) Starting from the mechanical energy balance

$$\frac{d}{dt} \int_{V(t)} \frac{\rho \mathbf{v} \cdot \mathbf{v}}{2} dV = \int_{S(t)} \mathbf{t} \cdot \mathbf{v} dS + \int_{V(t)} [-\boldsymbol{\tau} : \mathbf{D} + \rho \mathbf{b} \cdot \mathbf{v}] dV,$$

derive a conservation law of the form

$$\int_{V(t)} (\cdot) \rho dV = \text{constant}. \quad (1)$$

Now use Eqn. (1) and

$$\int_{V(t)} \frac{\rho \mathbf{v} \cdot \mathbf{v}}{2} dV = \frac{mL^3 \dot{\theta}^2}{6},$$

to derive the governing equation for  $\dot{\theta}$ , assuming that the rod is at rest at the initial position  $\theta = \theta_0$  in the accelerating frame of reference. State the initial condition for solving this equation. *Do not* attempt to solve this governing differential equation.

4. With  $\psi$  denoting the free energy, and  $\mathbf{T}$  denoting the first Piola-Kirchhoff stress tensor, the constitutive relations for a homogeneous solid are given by (25)

$$\begin{aligned} \psi &= \hat{\psi}(\mathbf{F}, \dot{\mathbf{F}}), \\ \mathbf{T} &= \hat{\mathbf{T}}(\mathbf{F}, \dot{\mathbf{F}}), \end{aligned}$$

where  $\mathbf{T}$  is a continuous function of  $\dot{\mathbf{F}}$ , and where a superposed dot indicates a material derivative.

- (a) Using the isothermal version of the Clausius-Duhem inequality given by

$$\left( \frac{\partial \hat{\psi}}{\partial t} \right)_X - \frac{1}{\rho_0} \mathbf{T} : \dot{\mathbf{F}} \leq 0,$$

find the equalities and inequalities (necessary and sufficient conditions) that result. In the proof for this part, *do not* attempt to construct an admissible thermodynamic process; assume that such a process can be constructed for arbitrary values of your kinematical variables. *Justify* all steps.

- (b) Apply the principle of material frame-indifference (MFI) to the reduced form of  $\hat{\psi}$  that you have derived in part (a). Next, using the chain rule, derive an expression for the part of the first Piola-Kirchhoff stress that depends on  $\hat{\psi}$ , so that MFI is automatically satisfied. State the condition for MFI for the remaining part of  $\mathbf{T}$ , but do not try to derive a reduced form in this case. You may directly use  $\mathbf{F}^* = \mathbf{QF}$ . Derive the required formulae for other kinematical variables.

## Some relevant formulae

$$\mathbf{b} = \mathbf{Q}^T [\mathbf{b}^* - \ddot{\mathbf{c}}] - \dot{\boldsymbol{\Omega}} \times \mathbf{x} - \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{x}) - 2\boldsymbol{\Omega} \times \mathbf{v}.$$

$$Q_{ij} = \mathbf{e}_i^* \cdot \mathbf{e}_j,$$

$$\boldsymbol{\Omega} = \begin{bmatrix} \dot{\mathbf{e}}_2 \cdot \mathbf{e}_3 \\ \dot{\mathbf{e}}_3 \cdot \mathbf{e}_1 \\ \dot{\mathbf{e}}_1 \cdot \mathbf{e}_2 \end{bmatrix},$$

$$\mathbf{R} : (\mathbf{S}\mathbf{T}) = (\mathbf{S}^T \mathbf{R}) : \mathbf{T} = (\mathbf{R}\mathbf{T}^T) : \mathbf{S} = (\mathbf{T}\mathbf{R}^T) : \mathbf{S}^T,$$

$$F_{iJ} = \frac{h_i}{h_J} \frac{\partial \hat{\chi}_i}{\partial \eta_J}, \quad \text{no sum on } i, J, \quad h_i \equiv (1, r, 1), \quad h_J \equiv (1, R, 1).$$

If  $\boldsymbol{\tau}$  is symmetric tensor-valued field, then the components of  $\nabla_x \cdot \boldsymbol{\tau}$  with respect to a cylindrical coordinate system are

$$(\nabla \cdot \boldsymbol{\tau})_r = \frac{\partial \tau_{rr}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{r\theta}}{\partial \theta} + \frac{\partial \tau_{rz}}{\partial z} + \frac{\tau_{rr} - \tau_{\theta\theta}}{r},$$

$$(\nabla \cdot \boldsymbol{\tau})_\theta = \frac{\partial \tau_{\theta r}}{\partial r} + \frac{1}{r} \frac{\partial \tau_{\theta\theta}}{\partial \theta} + \frac{\partial \tau_{\theta z}}{\partial z} + \frac{2\tau_{r\theta}}{r}.$$

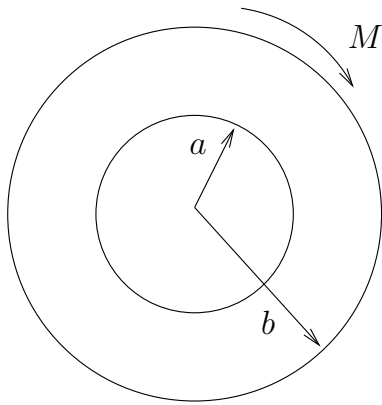


Figure 1: Hollow cylinder fixed rigidly at the inner boundary  $r = a$ , and subjected to a moment  $M$  on the outer boundary  $r = b$ .

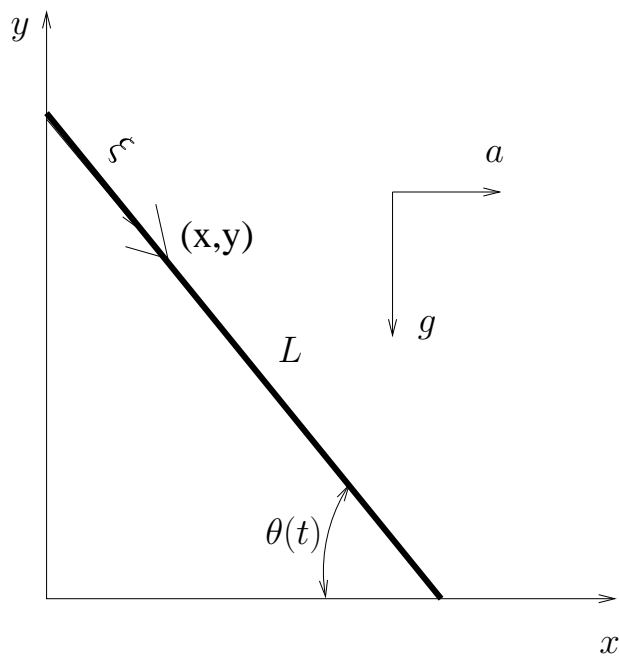


Figure 2: Problem 3.