

# Indian Institute of Science, Bangalore

## ME 257: Midsemester Test

**Date:** 16/2/2025.

**Duration:** 9.30 a.m.–11.30 a.m.

**Maximum Marks:** 100

1. Let the governing equation on a domain  $\Omega$  be given by (50)

$$\nabla \times \mathbf{T} + \mathbf{F} = \mathbf{0}, \quad (1)$$

where  $\mathbf{T}$  and  $\mathbf{F}$  are second-order tensors, and

$$(\nabla \times \mathbf{T})_{ij} = \epsilon_{irs} \frac{\partial T_{js}}{\partial x_r}.$$

Let the constitutive relation be given by

$$\mathbf{T} = E \nabla \times \boldsymbol{\epsilon}, \quad (2)$$

where  $E$  is a constant, and  $\boldsymbol{\epsilon}$  is a symmetric second-order tensor. By using a symmetric second-order tensor as a weighing function, and using indicial notation, develop the variational formulation corresponding to Eqn. (1), and derive the admissible kinematic and natural boundary conditions on the boundary  $\Gamma$  of the domain  $\Omega$ . In your final expression, the integrals on  $\Omega$  should be converted from indicial to tensorial notation, but the integrals on  $\Gamma$  *can be in indicial notation*. Thus, the natural boundary condition can also be expressed in terms of indicial notation without converting to tensorial notation (Hint: The bilinear term on  $\Omega$  in your variational formulation may or may not be symmetric).

2. Assume the body forces to be zero in this problem. A thick-walled hollow sphere (50) of inner radius  $a$  and outer radius  $b$  is subjected to a uniform pressure load  $p$  at its inner surface  $r = a$  as shown in Fig. 1. The outer surface  $r = b$  is in contact with a smooth rigid surface so that it cannot move radially outward. Our goal is to find the displacement and stress distribution in the sphere as a function of the spherical coordinates  $r$ - $\theta$ - $\phi$  where  $\theta \in [0, \pi]$  and  $\phi \in [0, 2\pi]$ .

- (a) Make suitable assumptions about the nature of the displacement field taking into account the symmetry of the structure, loading etc. (e.g.,  $u_\phi = 0$ ; in this step *do not* attempt to find the exact solution since we want to find the solution using FEM), simplify the following strain-displacement relations:

$$\begin{aligned} \epsilon_{rr} &= \frac{\partial u_r}{\partial r}, & \epsilon_{r\theta} &= \frac{1}{2} \left[ \frac{1}{r} \frac{\partial u_r}{\partial \theta} + r \frac{\partial}{\partial r} \left( \frac{u_\theta}{r} \right) \right], \\ \epsilon_{\theta\theta} &= \frac{1}{r} \left( \frac{\partial u_\theta}{\partial \theta} + u_r \right), & \epsilon_{\theta\phi} &= \frac{1}{2} \left[ \frac{1}{r \sin \theta} \frac{\partial u_\theta}{\partial \phi} + \frac{\sin \theta}{r} \frac{\partial}{\partial \theta} \left( \frac{u_\phi}{\sin \theta} \right) \right], \\ \epsilon_{\phi\phi} &= \frac{1}{r \sin \theta} \frac{\partial u_\phi}{\partial \phi} + \frac{u_r}{r} + \frac{u_\theta \cot \theta}{r}, & \epsilon_{r\phi} &= \frac{1}{2} \left[ \frac{1}{r \sin \theta} \frac{\partial u_r}{\partial \phi} + r \frac{\partial}{\partial r} \left( \frac{u_\phi}{r} \right) \right]. \end{aligned}$$

(b) With  $\lambda, \mu$  denoting the Lamé parameters, and with

$$\mathbf{C}_1 = \begin{bmatrix} \lambda + 2\mu & \lambda & \lambda \\ \lambda & \lambda + 2\mu & \lambda \\ \lambda & \lambda & \lambda + 2\mu \end{bmatrix}; \quad \mathbf{C}_2 = \mu \mathbf{I}_{3 \times 3},$$

$$\boldsymbol{\tau}_c = \begin{bmatrix} \mathbf{C}_1 & \mathbf{0}_{3 \times 3} \\ \mathbf{0}_{3 \times 3} & \mathbf{C}_2 \end{bmatrix} \boldsymbol{\epsilon}_c,$$

specialize the variational formulation that we developed in class

$$\int_{\Omega} [\boldsymbol{\epsilon}_c(\mathbf{v})]^T \boldsymbol{\tau}_c d\Omega = \int_{\Gamma_t} \mathbf{v} \cdot \bar{\mathbf{t}} d\Gamma \quad \forall \mathbf{v}, \quad (3)$$

to the problem at hand, i.e., write this equation in terms of your nonzero displacement and stress components. You may take  $d\Omega = 2\pi r^2 \sin \theta dr d\theta$ , and  $d\Gamma = 2\pi R^2 \sin \theta d\theta$  for the surface of a sphere of radius  $R$ .

- (c) Discretize the nonzero displacement components in your formulation using one quadratic element with midnode at the center of the element.
- (d) Derive the strain-displacement matrix  $\mathbf{B}$  for this element.
- (e) State the element stiffness matrix  $\mathbf{K}$ , and the element load vector  $\mathbf{f}$  in terms of  $\mathbf{B}$  and the shape function matrix  $\mathbf{N}$ . *Do not* carry out the integrations in the expressions for  $\mathbf{K}$ , but state them with the proper integration limits in the natural coordinate system. Find an explicit expression for  $\mathbf{f}$  which includes unknown reactions if any.
- (f) Writing your global set of equations  $\mathbf{K}\mathbf{u} = \mathbf{f}$  with the elements of  $\mathbf{K}$  denoted by  $K_{11}, K_{12}$  etc., and the elements of  $\mathbf{f}$  as  $f_1, f_2$  etc., show how you incorporate the appropriate boundary conditions for this problem into these set of equations.

### Some Relevant Formulae

For a quadratic 1-D element of length  $L$  with midnode at the center:

$$N_1(\xi) = \frac{\xi(\xi - 1)}{2}, \quad N_2(\xi) = (1 - \xi^2), \quad N_3(\xi) = \frac{\xi(\xi + 1)}{2}, \quad dx = \frac{L}{2} d\xi,$$

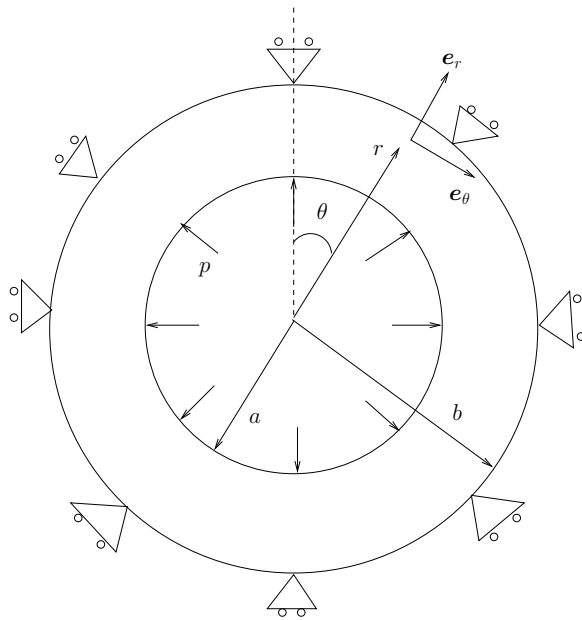


Figure 1: Problem 2.