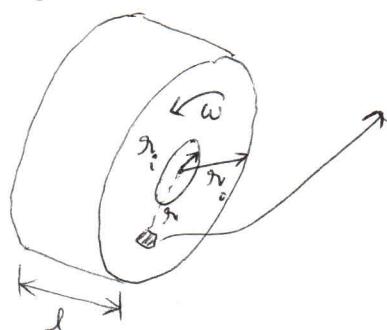


Thick-walled cylinders Revisited - Rotating disks

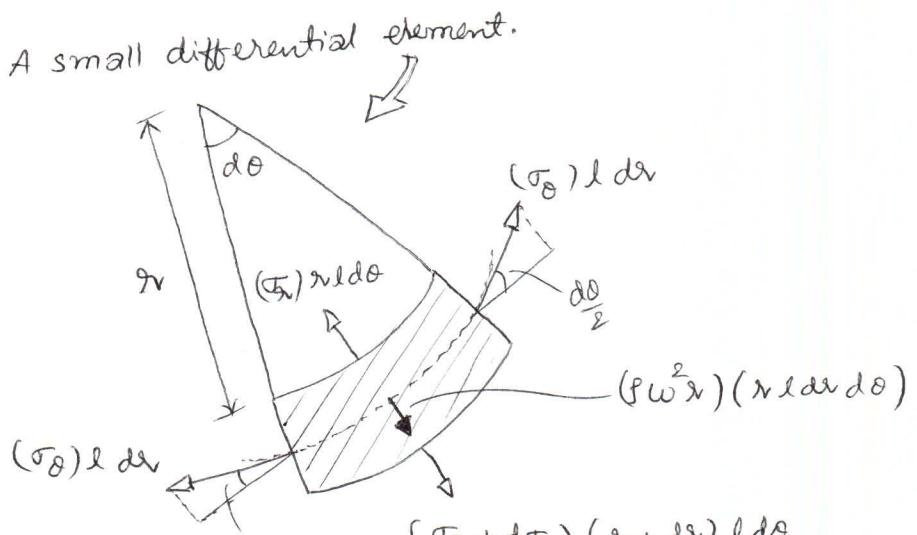
Previously, we had analyzed thick-walled cylinders as pressure vessels. Now, we consider a rotating disk such as a flywheel. In pressure vessels, we had internal and external fluid pressure on internal or external surfaces. But now, there is a centrifugal force acting everywhere as a "body-force". A body-force is a gravity-like force that acts at every point in the body. It should be noted that the pressure force acts only on the surface and so it is used as a boundary condition when we solve for displacements, strains, and stresses. A body-force, on the other hand, enters the differential equation directly. Let us see how...



w = angular velocity

ρ = mass density
 $= \text{kg/m}^3$

$\rho w^2 r$ = centrifugal force per unit volume



Balance forces in the radial direction:

$$\begin{aligned}
 & (\sigma_r + d\sigma_r)(r + dr)l d\theta + \rho w^2 r l dr d\theta = \sigma_r r l d\theta + 2\sigma_r l dr \sin \frac{d\theta}{2} \\
 \Rightarrow & \cancel{\sigma_r r l d\theta} + l \sigma_r dr d\theta + r l dr \cancel{d\theta} + l dr \cancel{d\theta} + \rho w^2 r l dr d\theta \\
 & = \sigma_r r l d\theta + \sigma_r l dr d\theta \\
 \Rightarrow & \left(\frac{d\sigma_r}{dr} + \frac{\sigma_r - \sigma_\theta}{r} + \rho w^2 r \right) (l dr d\theta) = 0
 \end{aligned}$$

$$\Rightarrow \frac{d\sigma_r}{dr} + \frac{\sigma_r - \sigma_\theta}{r} + 3\omega^2 r = 0$$

Recall that $\epsilon_r = \frac{du}{dr}$ { from $\frac{\text{change in length}}{\text{original length}}$ concept. }
 $\epsilon_\theta = \frac{u}{r}$

and $\begin{Bmatrix} \sigma_r \\ \sigma_\theta \end{Bmatrix} = \frac{E}{1-\nu^2} \begin{bmatrix} 1 & \nu \\ \nu & 1 \end{bmatrix} \begin{Bmatrix} \epsilon_r \\ \epsilon_\theta \end{Bmatrix}$ { from Hooke's law }

By substituting

$$\sigma_r = \frac{E}{1-\nu^2} (\epsilon_r + \nu \epsilon_\theta) = \frac{E}{1-\nu^2} \left(\frac{du}{dr} + \nu \frac{u}{r} \right)$$

$$\sigma_\theta = \frac{E}{1-\nu^2} (\nu \epsilon_r + \epsilon_\theta) = \frac{E}{1-\nu^2} \left(\nu \frac{du}{dr} + \frac{u}{r} \right)$$

$$\frac{d\sigma_r}{dr} = \frac{E}{1-\nu^2} \left(\frac{d^2 u}{dr^2} + \frac{\nu}{r} \frac{du}{dr} - \frac{\nu u}{r^2} \right)$$

into $\frac{d\sigma_r}{dr} + \frac{\sigma_r - \sigma_\theta}{r} + 3\omega^2 r = 0$, we get:

$$\boxed{\frac{d^2 u}{dr^2} + \frac{1}{r} \frac{du}{dr} - \frac{u}{r^2} + \frac{1-\nu^2}{E} + 3\omega^2 r = 0}$$

Governing differential equation

This has the following soln.

$$u = C_1 r + \frac{C_2}{r} - \frac{1-\nu^2}{E} \frac{3\omega^2 r^3}{8}$$

{ verify by substitution if you wish. }

C_1 and C_2 are constants to be evaluated from

the boundary conditions: 1) $\sigma_r = 0$ @ $r = r_i$
2) $\sigma_r = 0$ @ $r = r_o$.

Let us now use the boundary conditions to get c_1 and c_2 .

$$\sigma_r = \frac{E}{1-\nu^2} \left(\frac{du}{dr} + \nu \frac{u}{r} \right).$$

$$\begin{aligned} &= \frac{E}{1-\nu^2} \left(c_1 - \frac{c_2}{r^2} - \frac{1-\nu^2}{E} \frac{3}{8} 8\omega^2 r^2 + c_1 \nu + \frac{c_2 \nu}{r^2} \right. \\ &\quad \left. - \frac{1-\nu^2}{8E} 8\nu \omega^2 r^2 \right) \\ &= \frac{E}{1-\nu^2} \left((1+\nu) c_1 + \frac{c_2}{r^2} (\nu-1) - \frac{(1-\nu^2) 8\omega^2 r^2 (3+\nu)}{8E} (3+\nu) \right) \end{aligned}$$

$$\sigma_r = 0 \quad \text{for } r = r_i \quad \text{and} \quad \text{for } r = r_o.$$

$$\Rightarrow (1+\nu) c_1 + \frac{c_2}{r_i^2} (\nu-1) = \frac{(1-\nu^2) 8\omega^2 r_i^2 (3+\nu)}{8E}$$

$$\text{and } (1+\nu) c_1 + \frac{c_2}{r_o^2} (\nu-1) = \frac{(1-\nu^2) 8\omega^2 r_o^2 (3+\nu)}{8E}$$

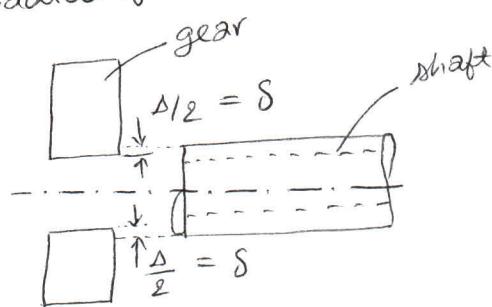
Solve for c_1 and c_2 and substitute into σ_r and σ_θ

$$\sigma_r = \frac{3+\nu}{8} 8\omega^2 \left(r_o^2 + r_i^2 - \frac{r_o^2 r_i^2}{r^2} - \frac{r^2}{r^2} \right)$$

$$\sigma_\theta = \frac{3+\nu}{8} 8\omega^2 \left(r_o^2 + r_i^2 + \frac{r_o^2 r_i^2}{r^2} - \frac{1+3\nu}{3+\nu} \frac{r^2}{r^2} \right).$$

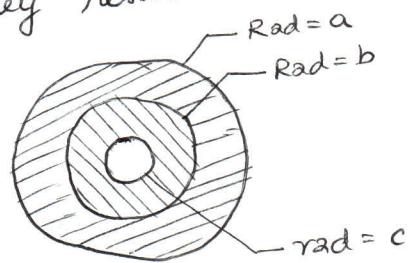
To really understand this solution, we should consider one more application of this, namely, "interference fits".

Let us say that we want to mount a gear on a hollow shaft by pressing the gear onto the shaft with force. It is called an "interference fit" or a "press fit". For this to work, the inner radius of the gear must be a little smaller than the outer radius of the shaft. See below.

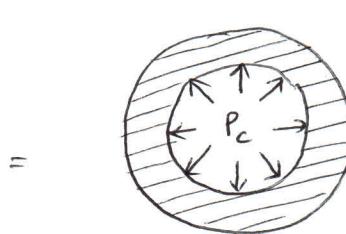


$$\begin{aligned}s &= \text{radial interference} \\ &= r_{\text{shaft}} - r_{\text{gear}} \\ &\quad (\text{out}) \quad (\text{in})\end{aligned}$$

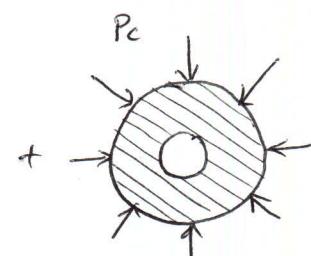
when we do this press-fit, there will be an "interface" pressure between the gear and the shaft. Let us call it P_c . P_c is like an internal pressure on the gear and makes it expand some. P_c is like an external pressure on the shaft and makes it contract some. Together, they result in s so that they fit together nicely.



After press fit.



Get u_x _{gear}



Get u_x _{shaft}

$$s = (u_x)_{\text{gear}} @ r = b - (u_x)_{\text{shaft}} @ r = b$$

Condition to find P_c for given s