

ME 254, Lecture 1

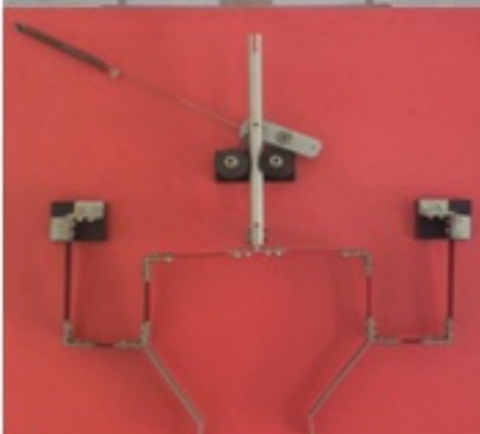
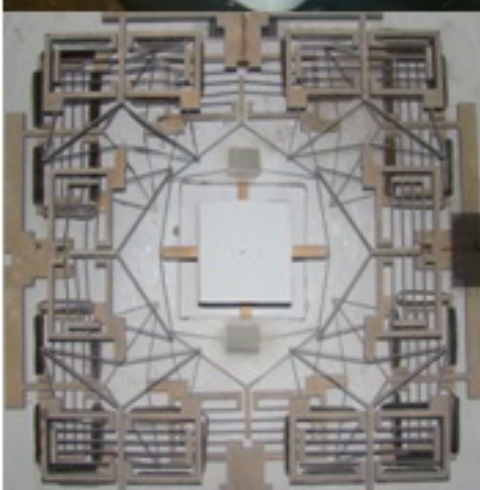
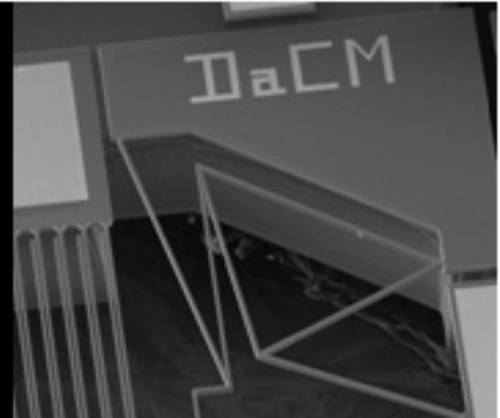
# Compliant mechanisms — an overview

G. K. Ananthasuresh  
[suresh@iisc.ac.in](mailto:suresh@iisc.ac.in)



# Compliant Mechanisms

macro . meso . micro . nano



# Compliant Mechanisms

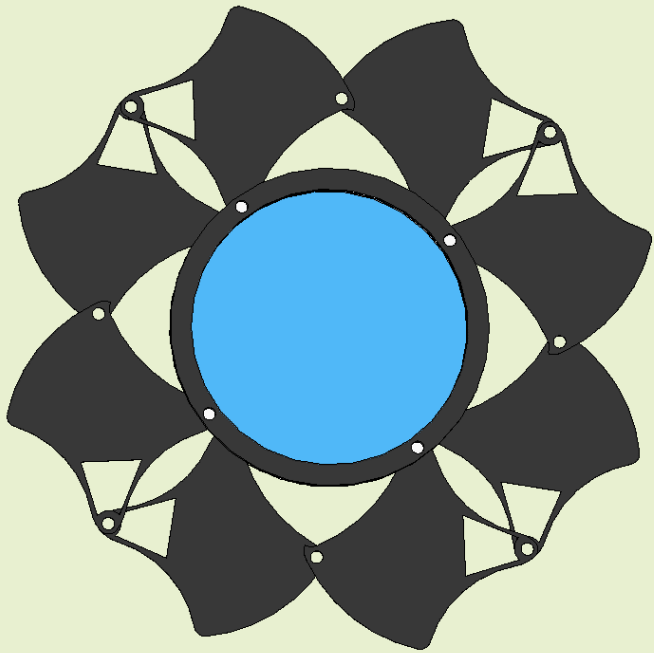
Motion without Hinges and Sliders



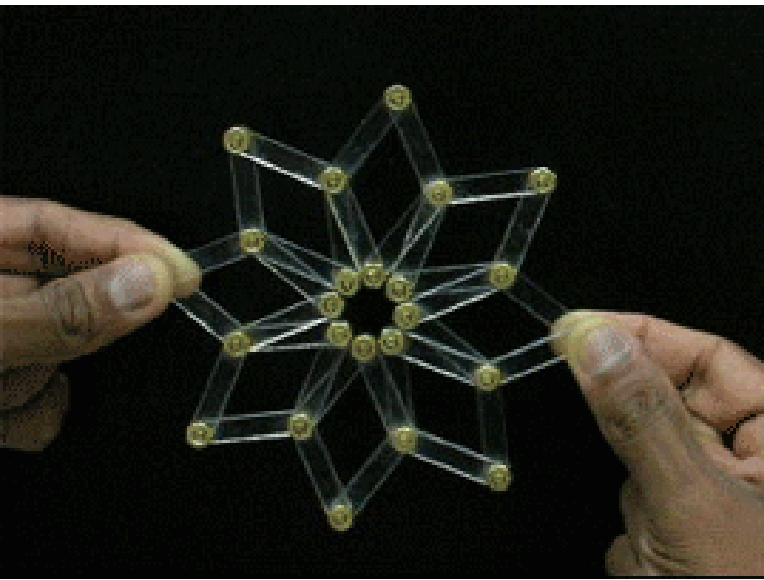
G.K. Ananthasuresh  
and

**All the current and past members of the  
M2D2 Laboratory**

Mechanical Engineering  
Indian Institute of Science  
Bangalore



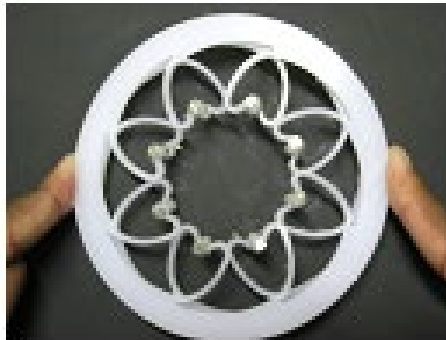
**Rigid-body mechanisms**



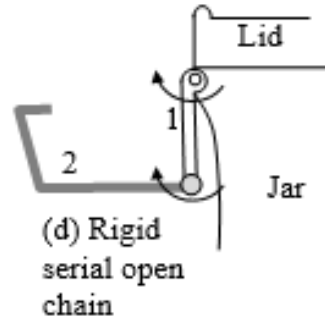
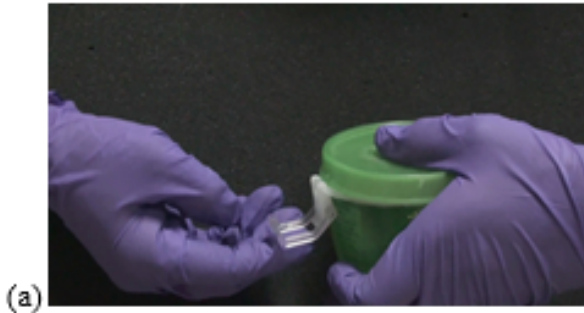
## Equivalent compliant mechanism

Int. J. Solids and Structures  
44(2007), pp. 6279-6298  
*With Jiten Patel*

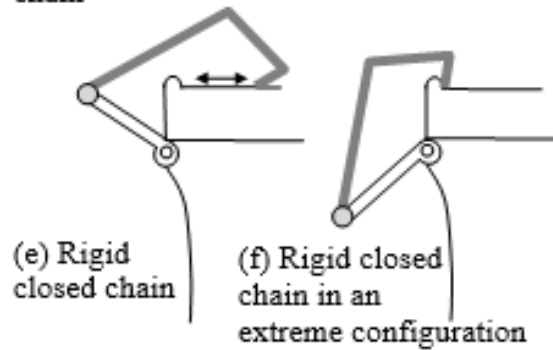
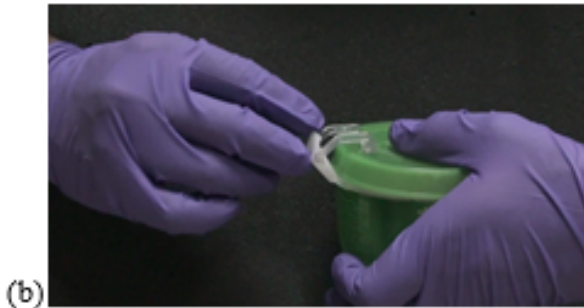
## Compliant mechanisms



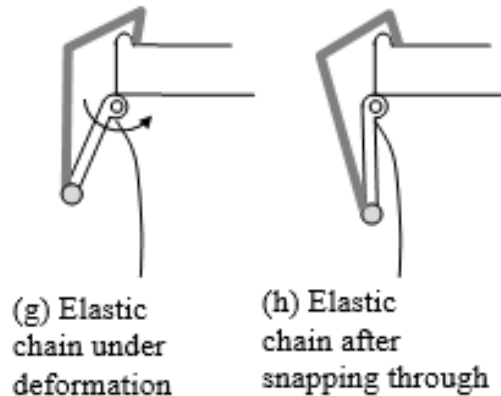
# Smooth transition of mobility



Rigid serial chain



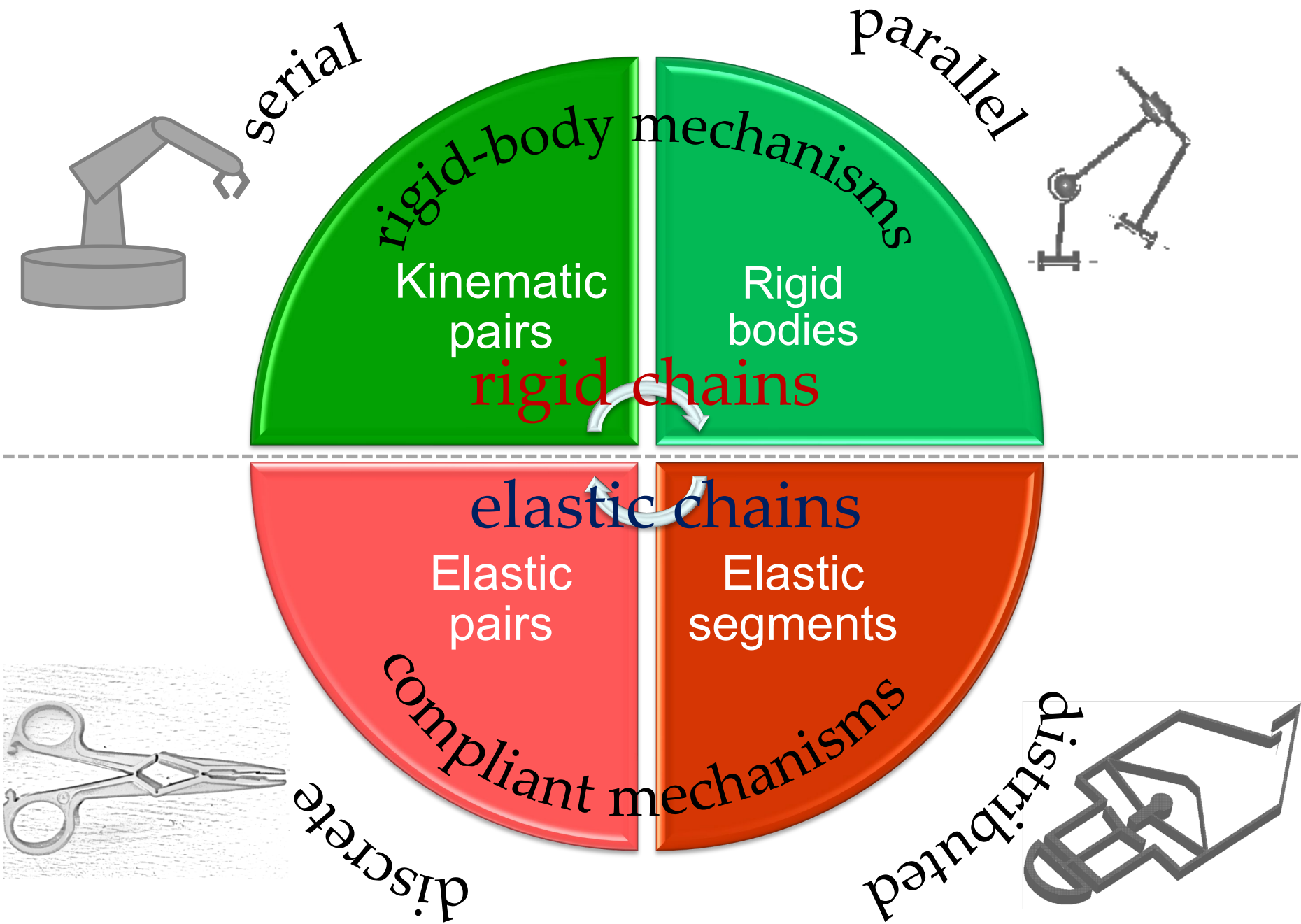
Rigid closed chain



Elastic chain

# BRU coffee-jar lid





# Kinematic pairs

- Lower pairs with surface contact
  - Revolute (hinge-1 DoF)
    - Surface of revolution
  - Prismatic (slider-1 DoF)
    - Prismatic surface
  - Helical (screw-1 DoF)
  - Cylindrical (2 DoF)
  - Toroidal (2 DoF)
  - Spherical (ball-socket-3 DoF)
  - Planar (3 DoF)
- And there are many, many **higher pairs** with line or point contact

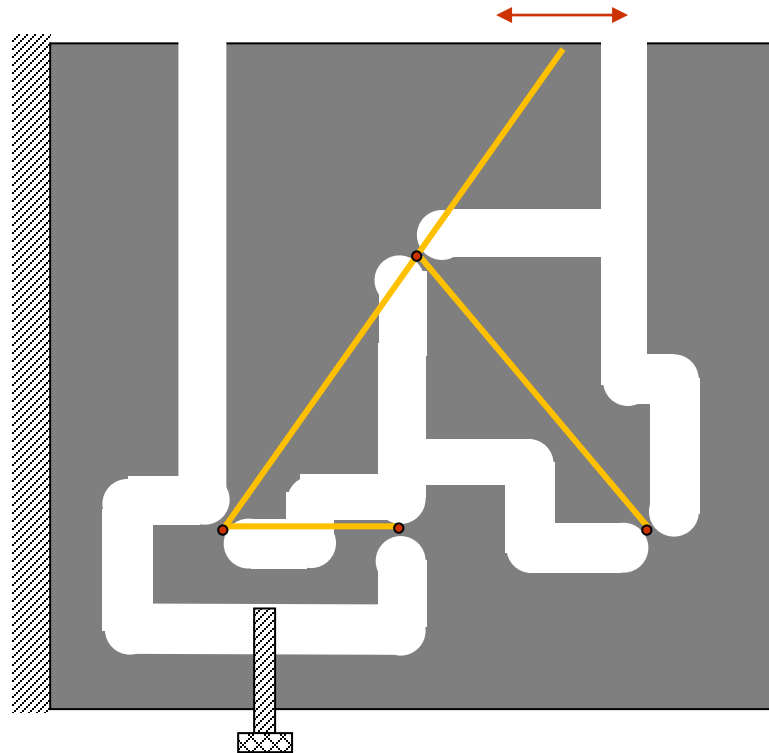
Specific shapes  
for contact  
surfaces



Prefer hinges to sliders;  
flexures to either.

A design principle espoused by M. J. French.

# Discrete compliance



# Elastic pairs

No specific  
shape

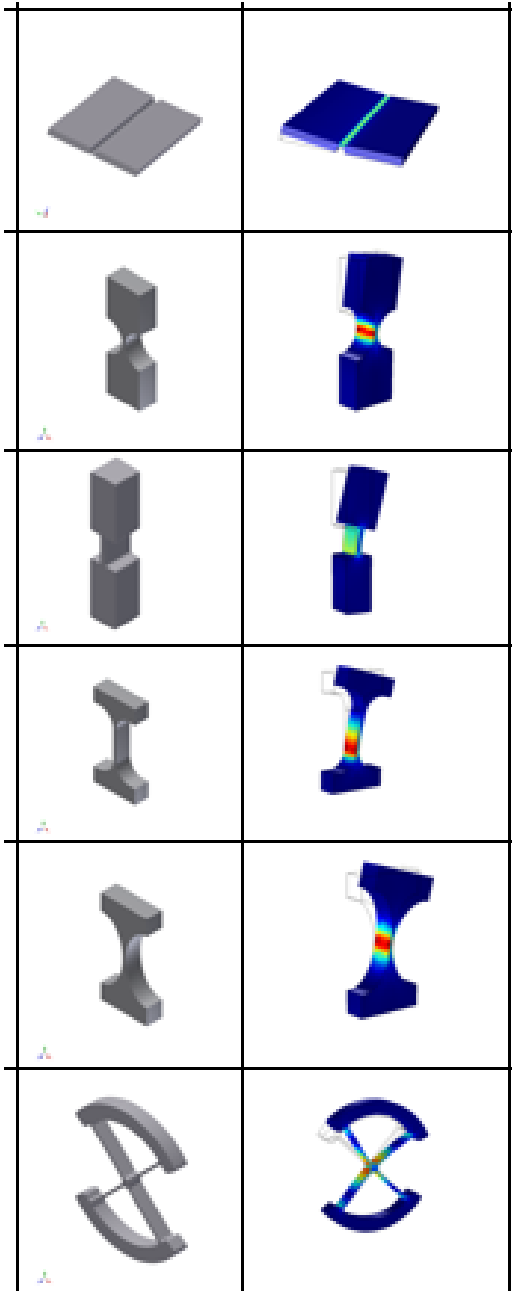
Elastic 2-DoF  
rotational  
pairs

Elastic 1-DoF  
rotational  
pairs

Elastic 1-DoF  
sliding pairs

With  
Ashwin Rao  
Santosh Bhargav

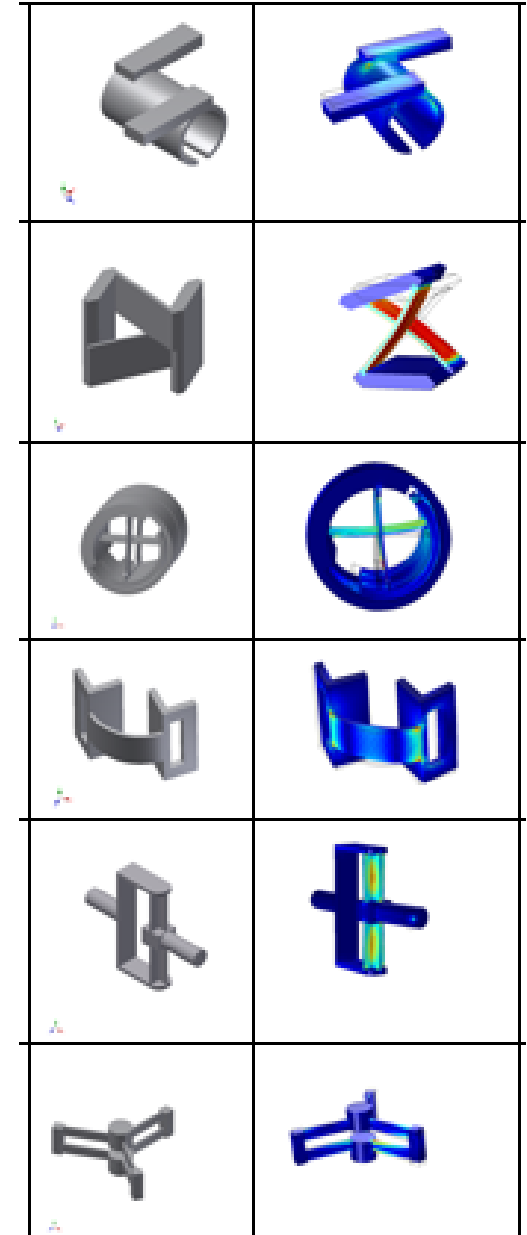




# 1-DoF rotational elastic pairs

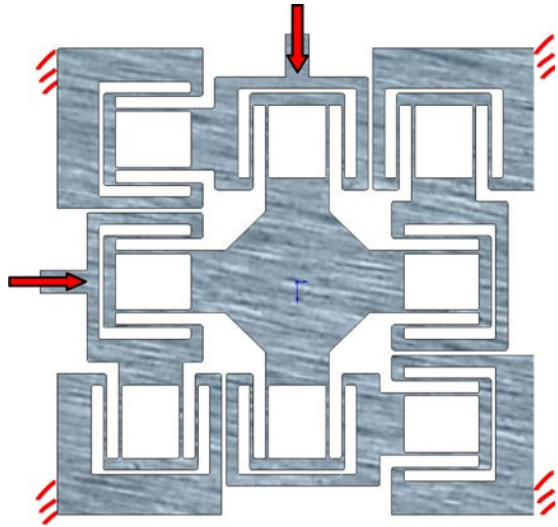
More than a dozen  
shapes for 1-DoF  
elastic rotational  
pairs!

Bendix elastic  
rotational pair

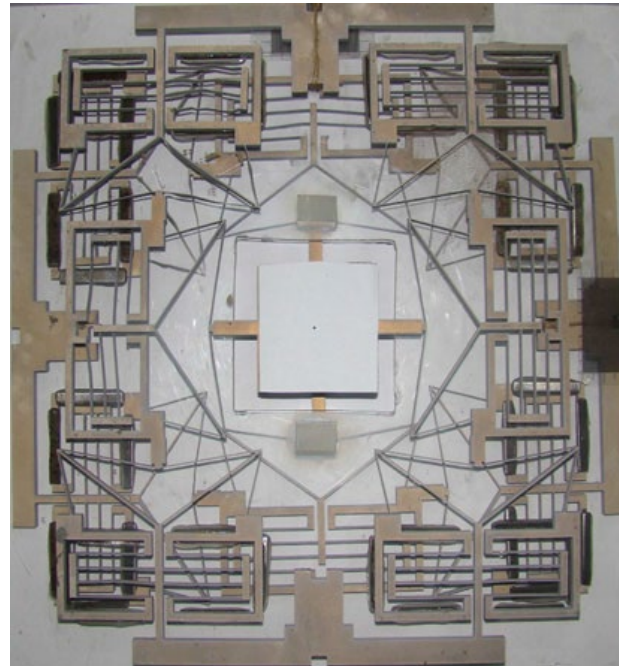


# Shapes of elastic pairs can be quite complicated.

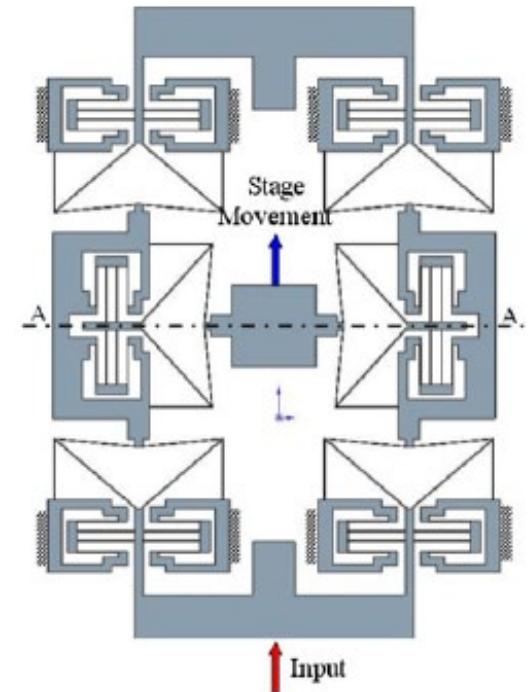
You can even call them compliant mechanisms in their own right!



Awtar, 2003



With Dinesh Mana



# Distinguishing the pairs quantitatively

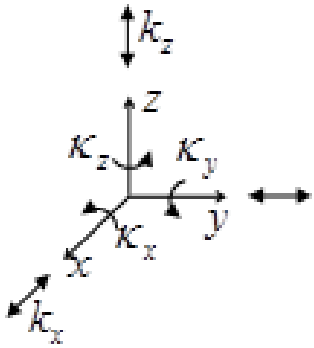
## Kinematic

- Ideally...
  - Zero stiffness along or about the intended axis.
  - Infinite stiffness along or about all other five axes.
  - Almost no cross-axis errors.
- Friction and backlash cause deviation from the ideal condition.

## Elastic

- Ideally and realistically...
  - Finite but low stiffness along or about the intended axis.
  - Finitely large stiffness along or about all other five axes.
  - Finite cross-axis errors
- Friction and backlash are absent.
- Viscoelastic behavior may cause deviations.
- Axis may drift.

# Multi-axis stiffness of an elastic pair



$$\mathbf{Ku} = \mathbf{f} \Rightarrow \begin{bmatrix} k_{xx} & k_{xy} & k_{xz} & k_{x\theta} & k_{x\phi} & k_{x\psi} \\ & k_{yy} & k_{yz} & k_{y\theta} & k_{y\phi} & k_{y\psi} \\ & & k_{zz} & k_{z\theta} & k_{z\phi} & k_{z\psi} \\ & & & k_{\theta\theta} & k_{\theta\phi} & k_{\theta\psi} \\ & & & & k_{\phi\phi} & k_{\phi\psi} \\ & & & & & k_{\psi\psi} \end{bmatrix} \begin{Bmatrix} u_x \\ u_y \\ u_z \\ \theta \\ \phi \\ \psi \end{Bmatrix} = \begin{Bmatrix} F_x \\ F_y \\ F_z \\ M_x \\ M_y \\ M_z \end{Bmatrix}$$

Symmetric

Elastic deformation analysis, analytical or numerical, via the compliance matrix can be used to compute  $\mathbf{K}$ .

# Computing the multi-axis compliance matrix

$$\mathbf{K}^{-1} = \mathbf{C} \Rightarrow \begin{bmatrix} c_{xx} & c_{xy} & c_{xz} & c_{x\theta} & c_{x\phi} & c_{x\psi} \\ & c_{yy} & c_{yz} & c_{y\theta} & c_{y\phi} & c_{y\psi} \\ & & c_{zz} & c_{z\theta} & c_{z\phi} & c_{z\psi} \\ & & & c_{\theta\theta} & c_{\theta\phi} & c_{\theta\psi} \\ & & & & c_{\phi\phi} & c_{\phi\psi} \\ & & & & & c_{\psi\psi} \end{bmatrix} \begin{Bmatrix} F_x \\ F_y \\ F_z \\ M_x \\ M_y \\ M_z \end{Bmatrix} = \begin{Bmatrix} u_x \\ u_y \\ u_z \\ \theta \\ \phi \\ \psi \end{Bmatrix}$$

*Symmetric*

Up to six analysis runs...

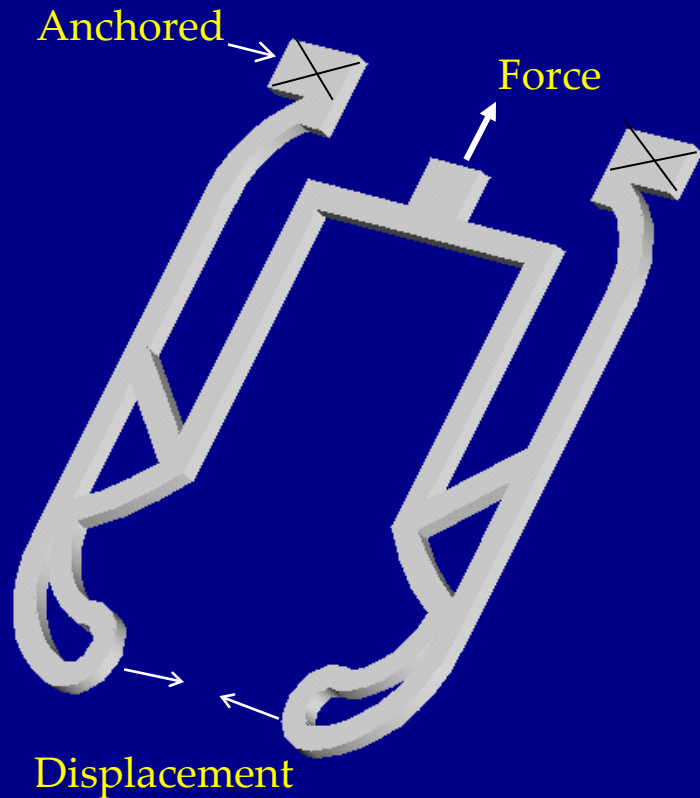
Three finite element analysis runs in 2D.

Six finite element analysis runs in 3D.



# No joints (pairs) at all

A compliant mechanism  
with **elastic segments**



**Elastic segments**  
instead of  
elastic pairs.

## Distributed compliance

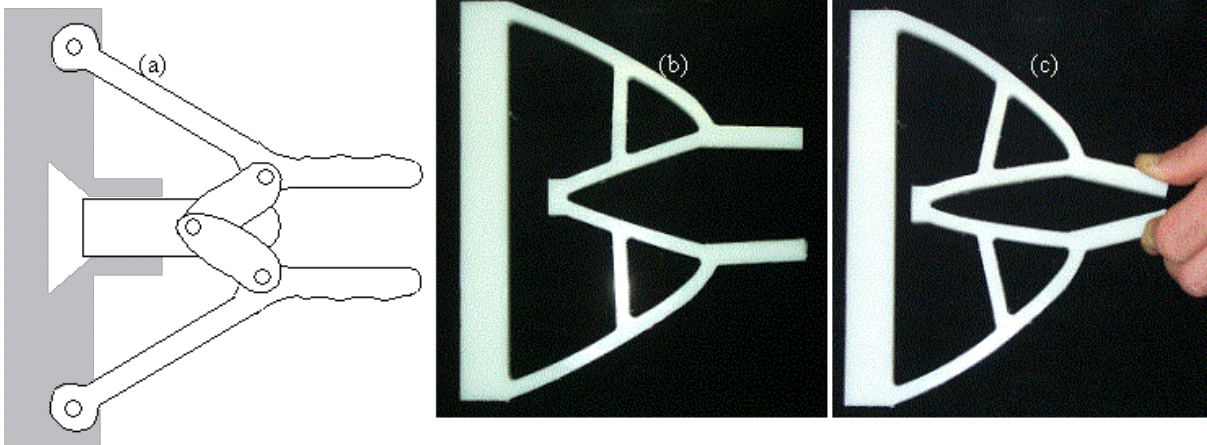
No elastic pairs.  
Uniformly distributed deformation.  
Large displacement with small strain.  
Stronger than elastic pairs.  
Enhanced scope for design.

# A compliant gripper



# Compliant Mechanism: the definition

- A mechanism that uses elastic deformation to **transmit** or **transform** force, motion, and energy.

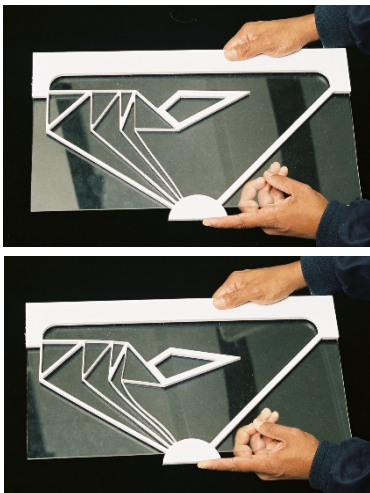
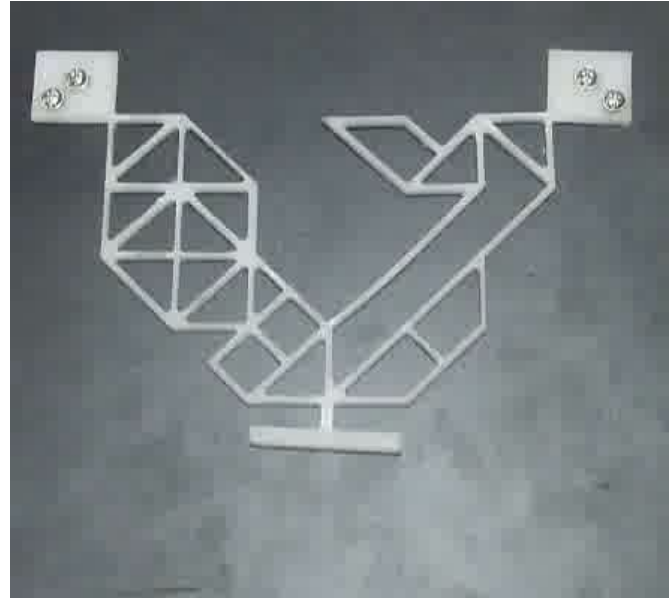


Replace many parts with one part.

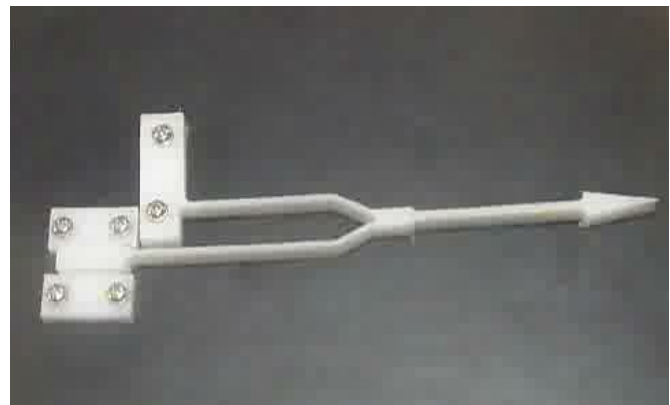
# Transmission

## Transmission

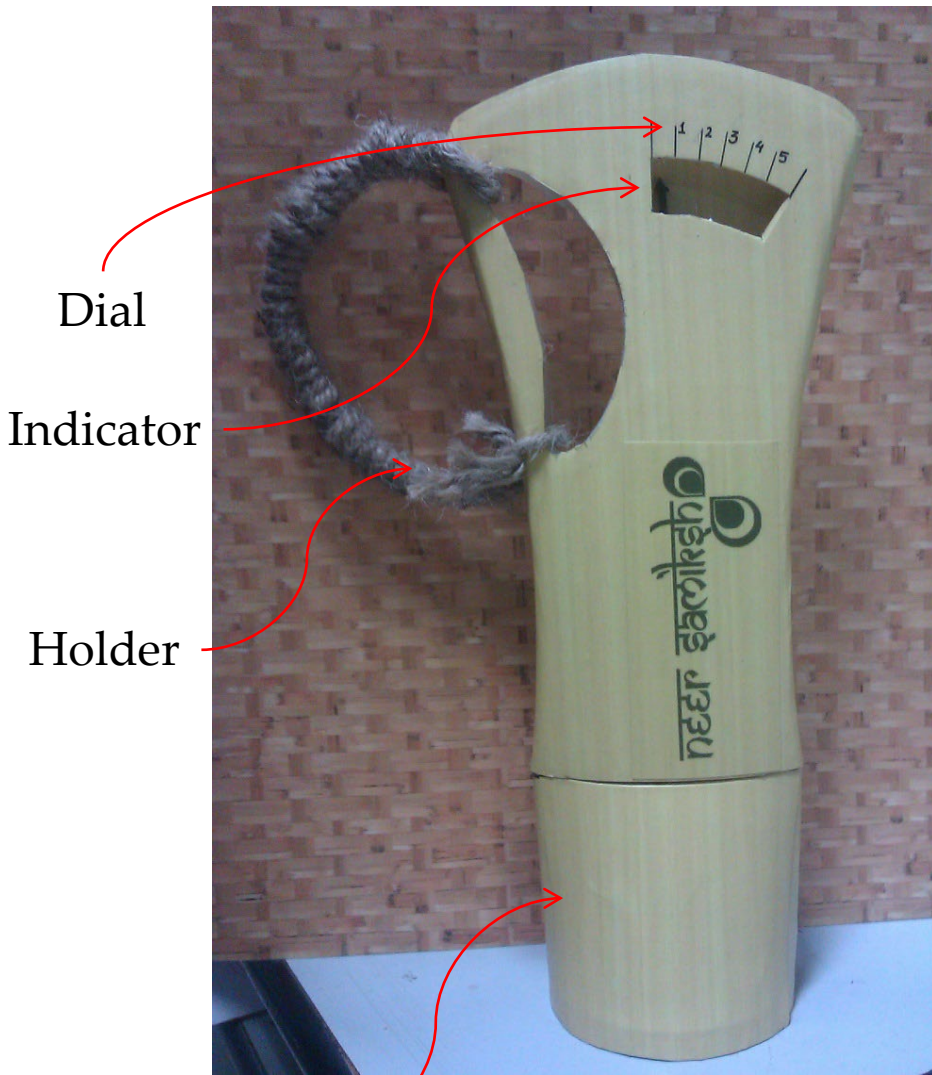
- Amplify force or motion
- Change direction
- Change the dynamics
- Change state



With  
Anupam  
Saxena and  
Luzhong  
Yin







Dial

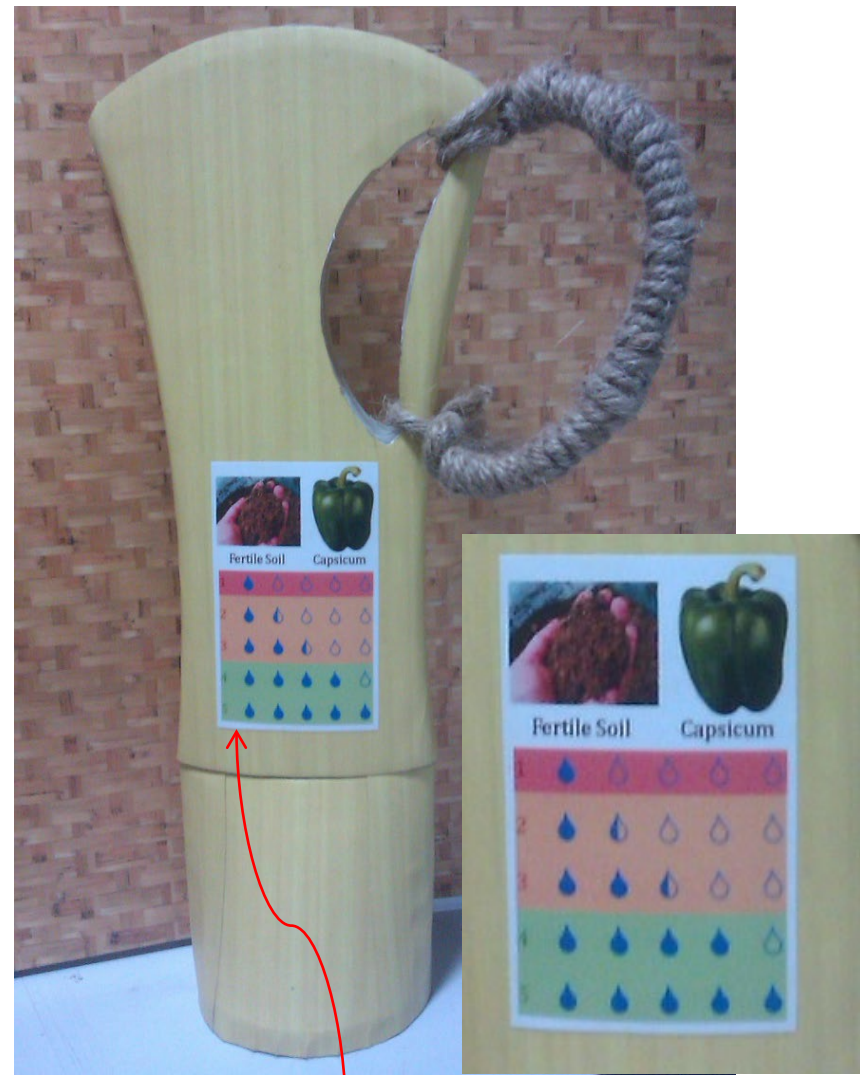
Indicator

Holder

Soil Sample  
Holder

# Neer Samiksh

Ananthasuresh, IISc



Suggestive remarks as per  
the crop and soil

# Soil-moisture sensor

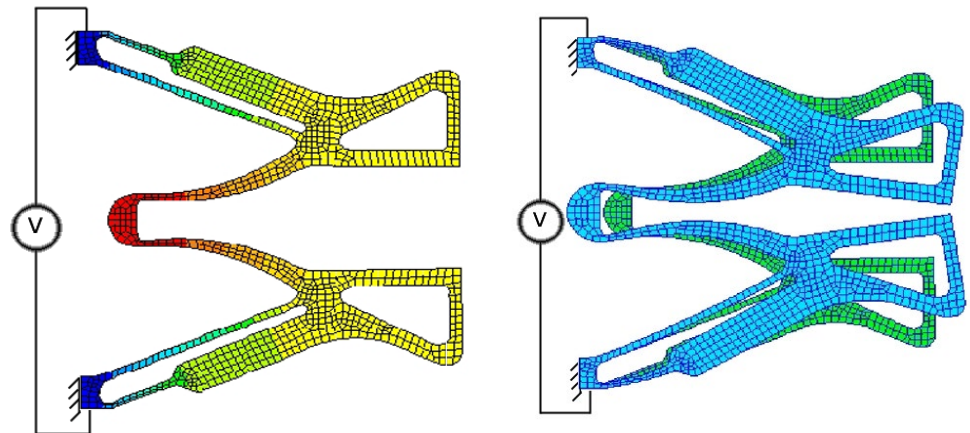


# Transmission is fine; *transformation?*



## Transduction

- Convert non-mechanical energy into mechanical energy and vice versa.
- Smart or active materials

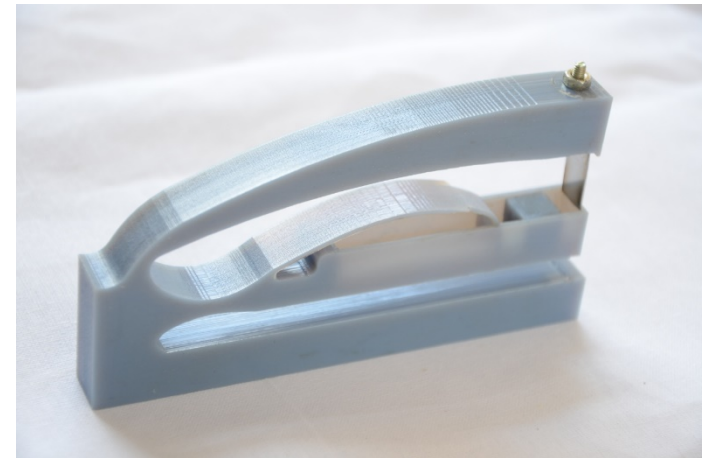
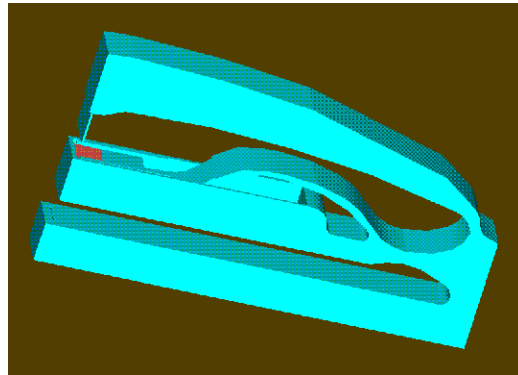


With Moulton



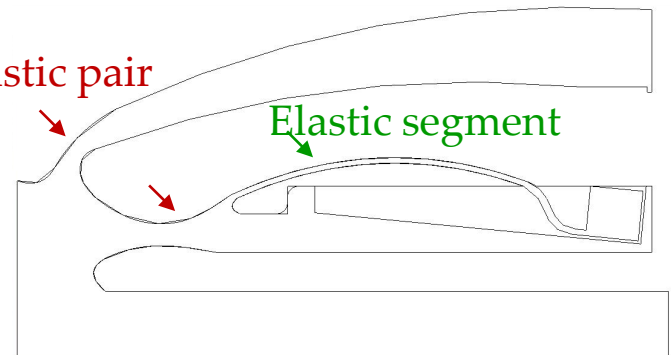
# Monolithic (uni-body; single-piece) construction

- Ease of manufacture – reduced or no assembly



Elastic pair

Elastic segment



Ananthasuresh and  
Saggere, 1994

# Monoform compliant designs from FlexSys



Shape-shifting Things to Come  
Sridhar Kota

May 2014, [ScientificAmerican.com](http://ScientificAmerican.com)

# One inside the other... multi-scale graspers



With  
Nandan Maheswari

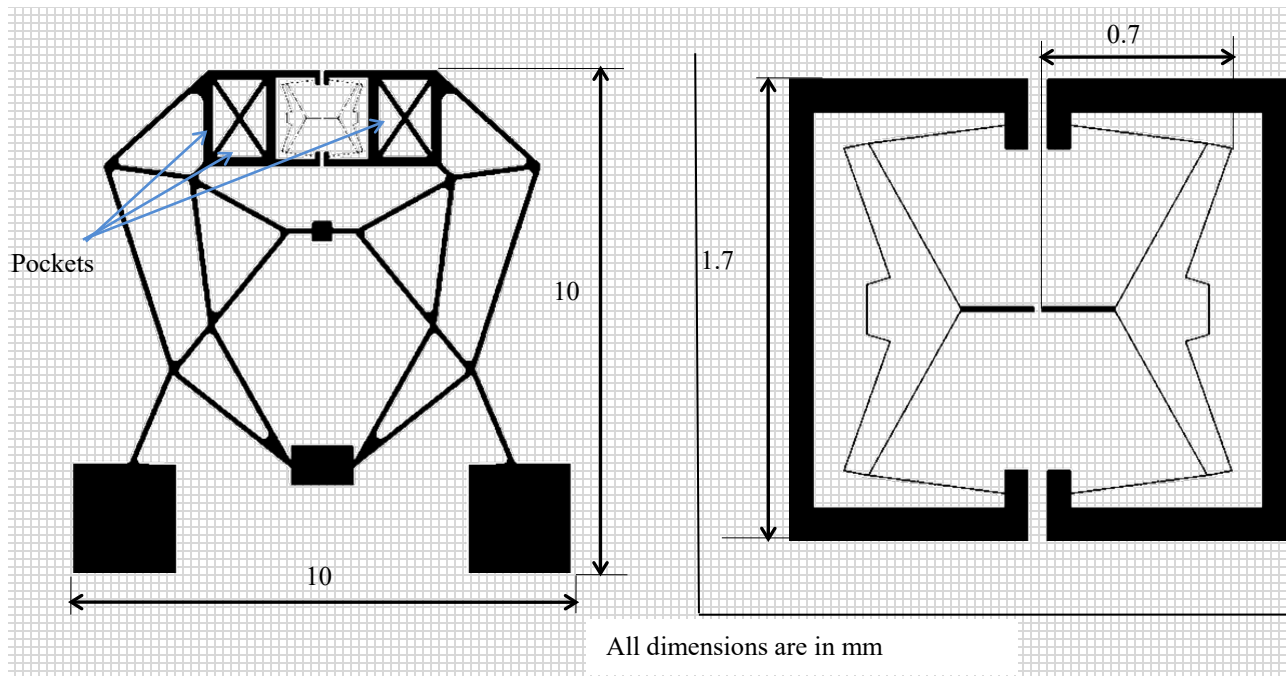
# Stiffness matched to that of the biological cells

Zebrafish egg squeezing

Zebrafish egg rolling

Too soft for drosophila embryo

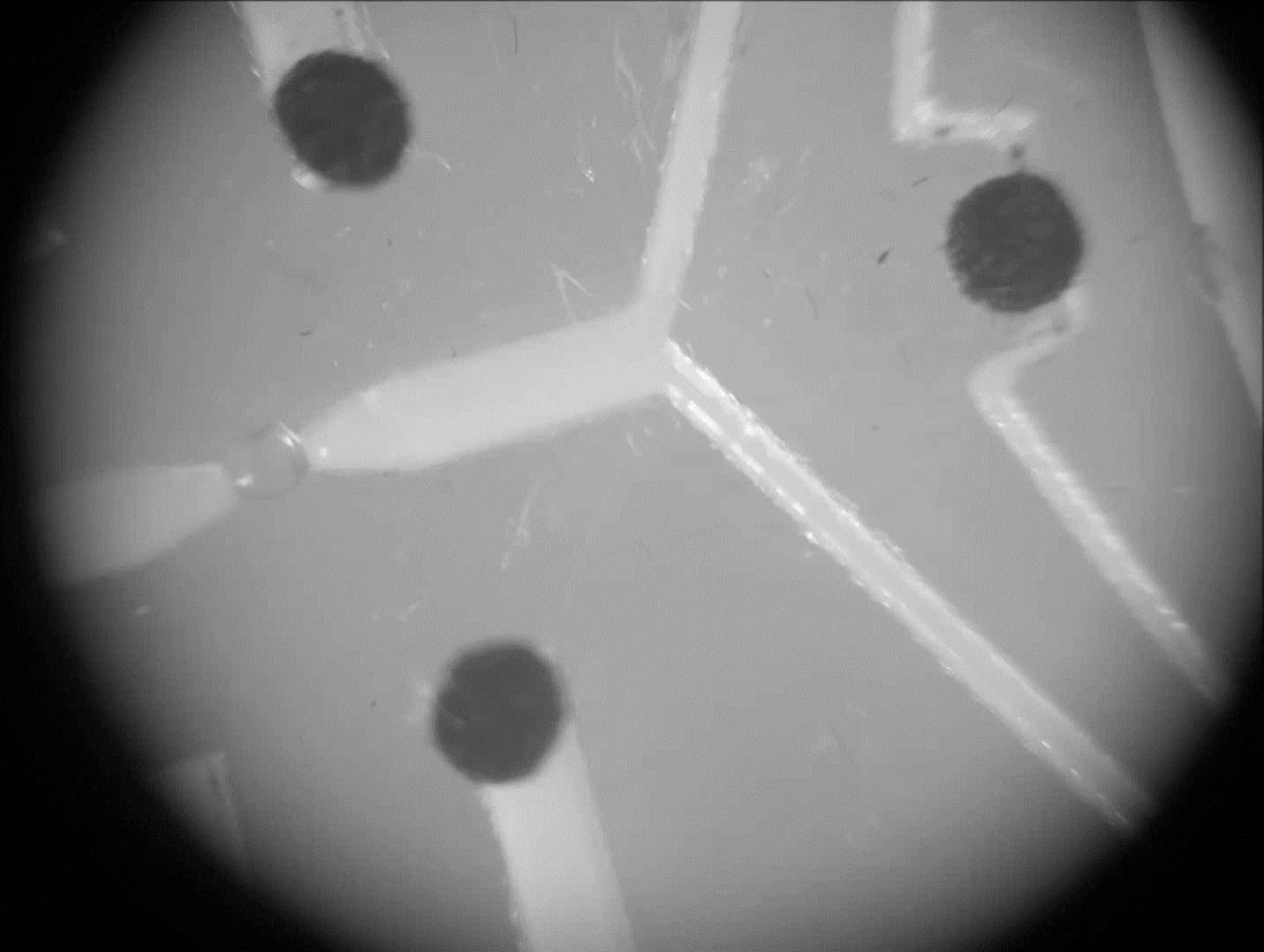
MCF-7 cell grasping



With  
Santosh  
Bhargav

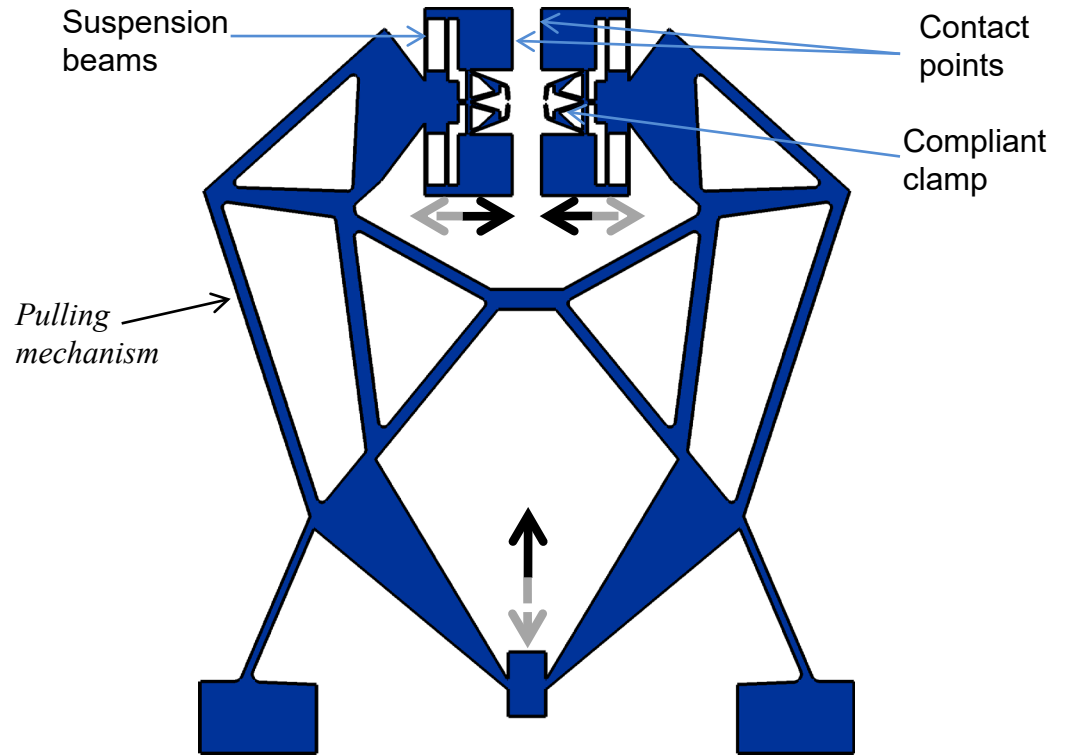
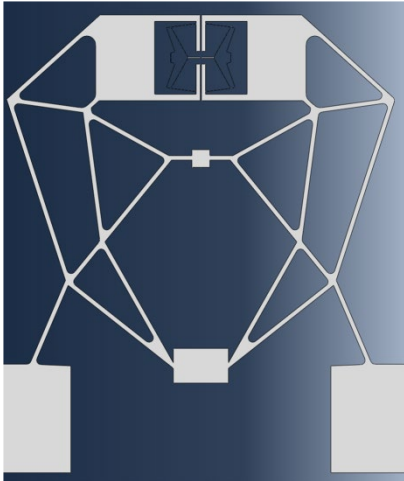
Force estimated to deform the [zebrafish embryo](#) by  $150 \mu\text{m}$  is  $1.2 \text{ mN}$ . Thus the bulk stiffness is about  $8 \text{ N/m}$

# Squeezing a zebrafish egg... for mechanical characterization



# Tuning stiffness... by combining

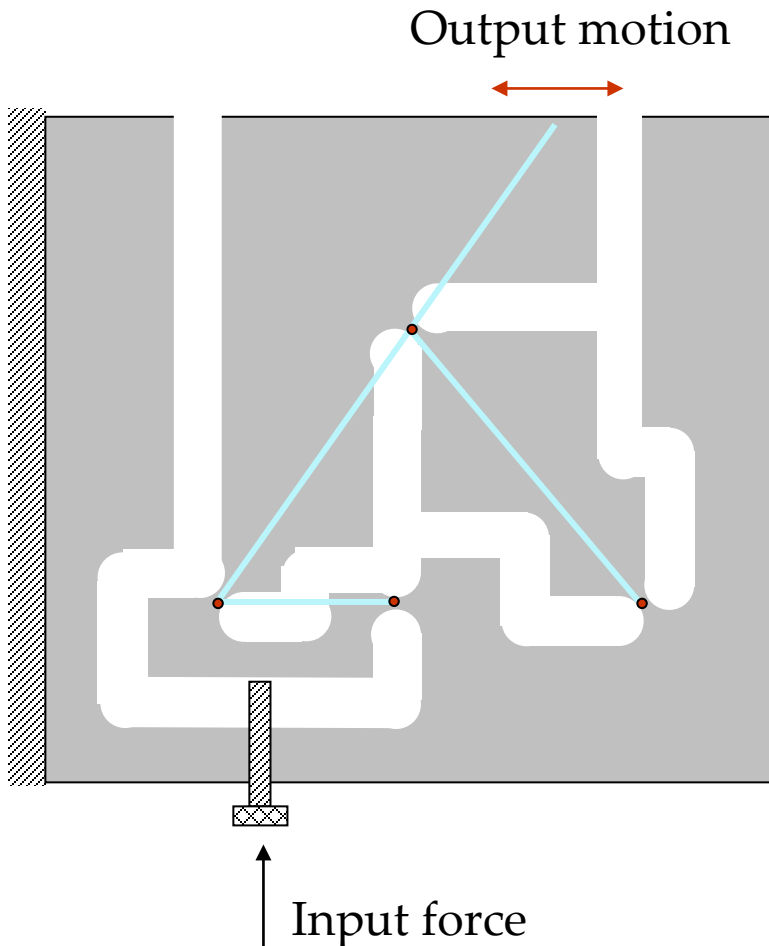
**Gripper + DaCM =>  
Stiffness tuning + GA**



Stretch too...

With  
Santosh  
Bhargav

# No more rubbing the wrong way



- Ease of manufacture – reduced or no assembly
- Ability to withstand overloads – “I bend but I break not.”
- Less or no friction and wear
- No backlash problem – more precision

Prefer hinges to sliders,  
flexures to either.  
- M. J. French

# Real applications



- Ease of manufacture – reduced or no assembly
- Ability to withstand overloads – “I bend but I break not.”
- Less or no friction and wear
- No backlash problem – more precision
- Aesthetics made easier
- **Economy of material and less cost for better performance**



Over-running and one-way clutches

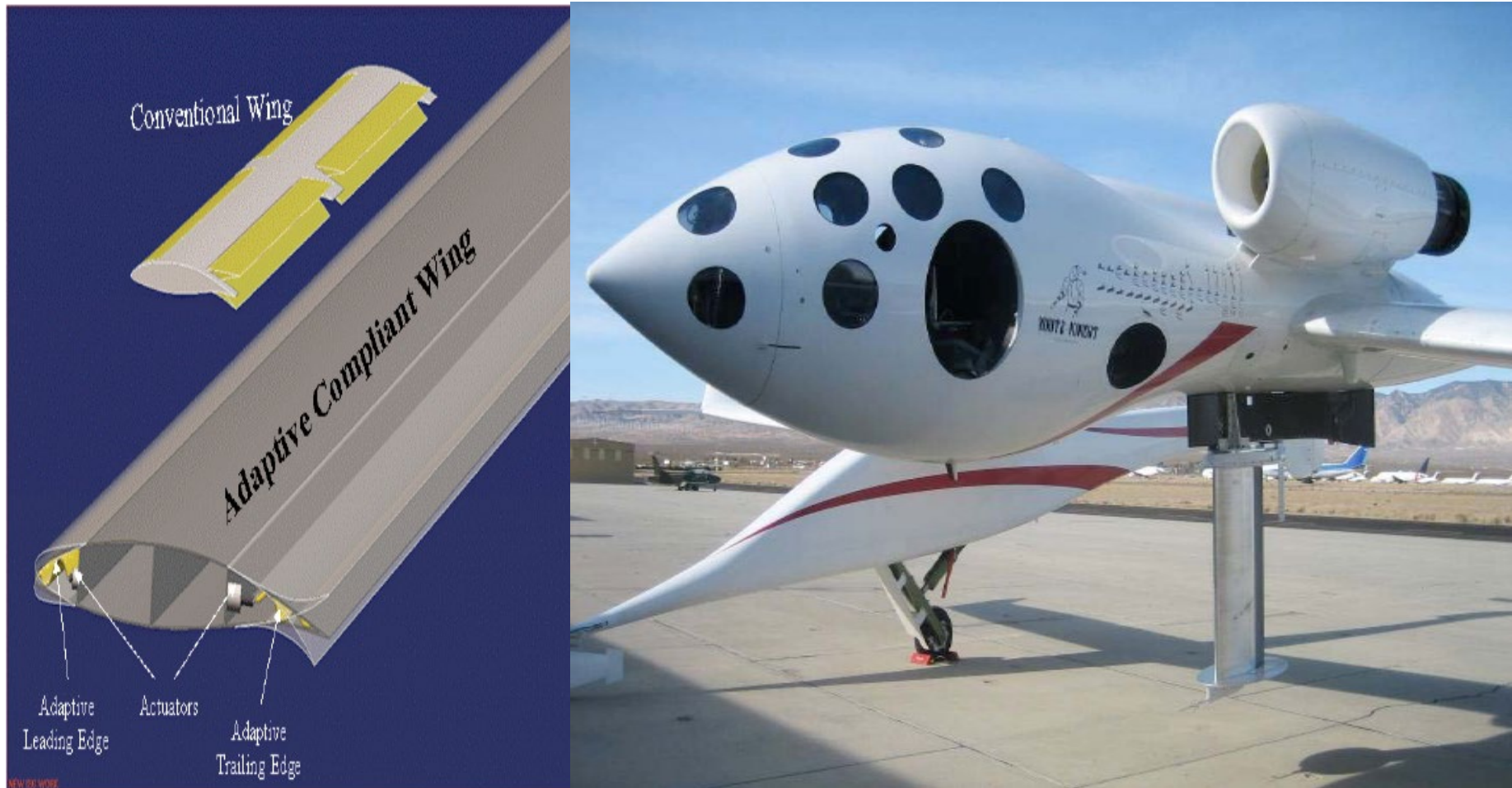


Prof. Larry Howell  
Brigham Young University

Gahring and Ananthasuresh



# Mission-adaptive compliant wings



Prof. Sridhar Kota, University of Michigan

Ananthasuresh, IISc

# Morphing wing surface

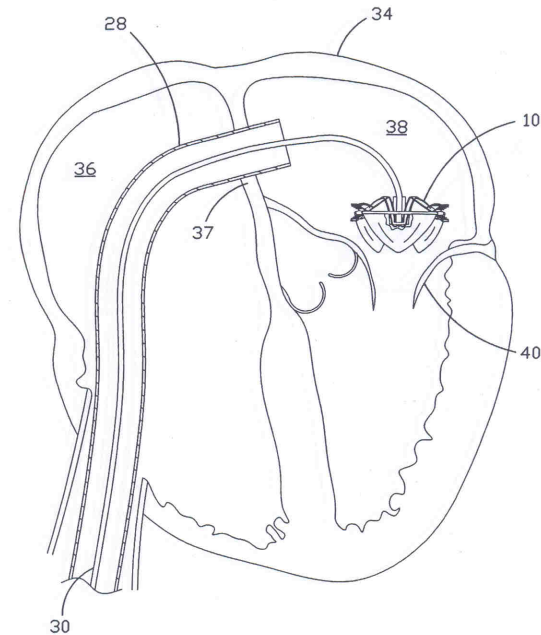
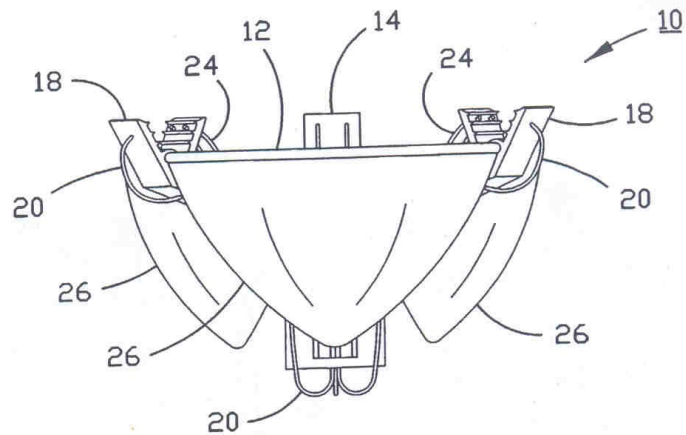


**WITH ELASTIC DESIGN**, flexible, morphing surfaces can replace rigid wing flaps (*r*), windshield wiper frames can be molded from single pieces of material (*s*), and one chunk of plastic can do the work of a conventional stapler's nearly two dozen parts (*x*).

Shape-shifting  
Things to Come  
Sridhar Kota

May 2014, [ScientificAmerican.com](http://ScientificAmerican.com)

# An artificial heart valve

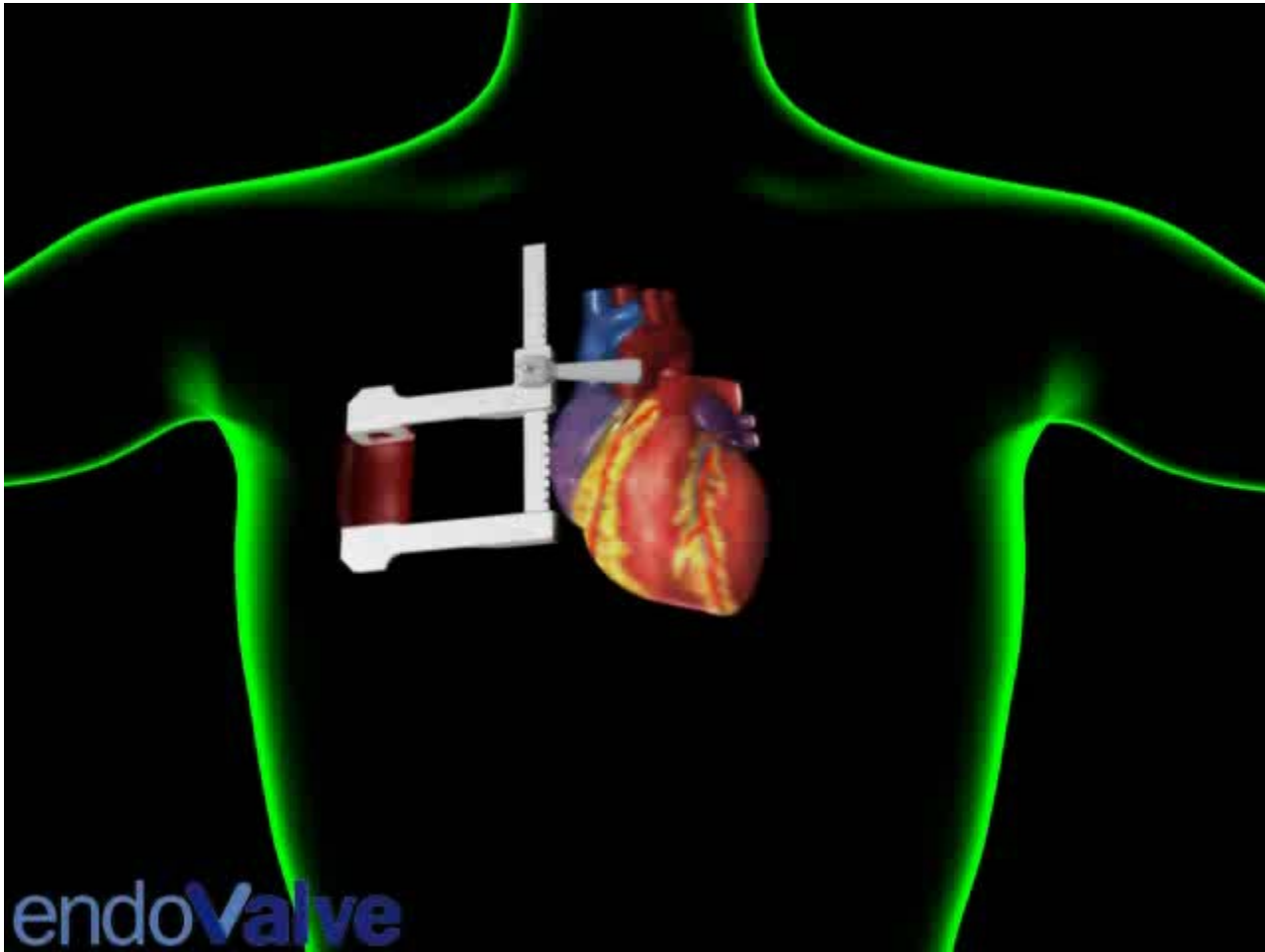


Currently under further development by Endo Valve, Inc., USA

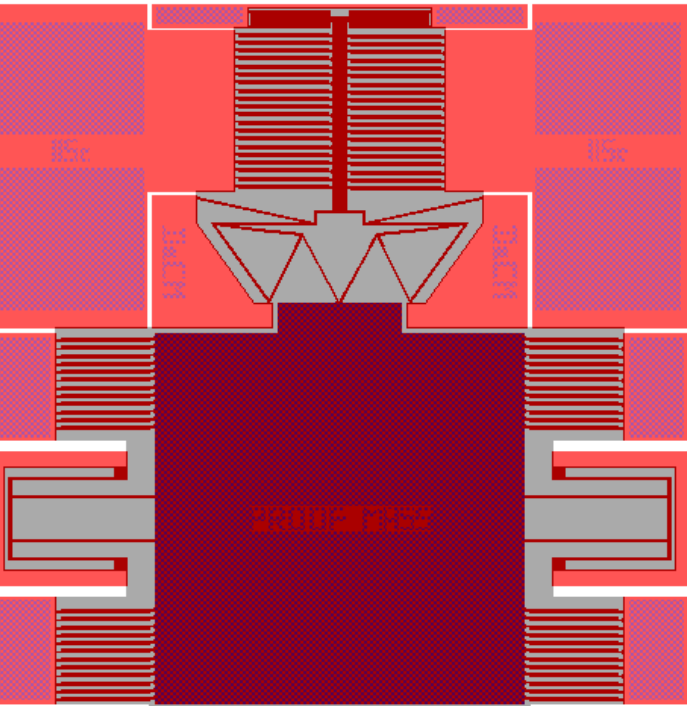
## Patent

“Percutaneous Heart Valve,” Patent application #.

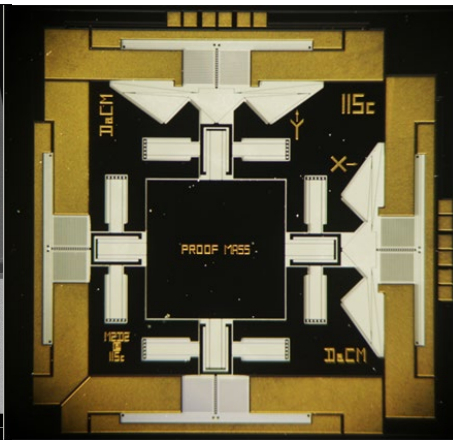
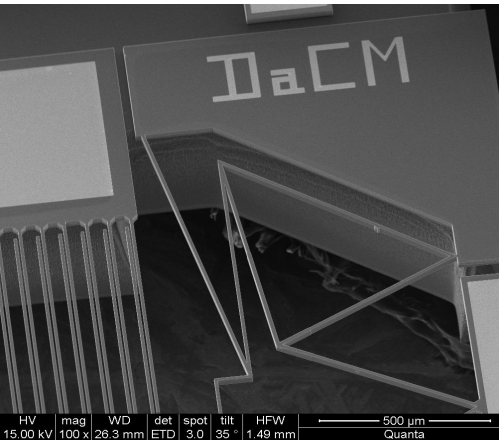
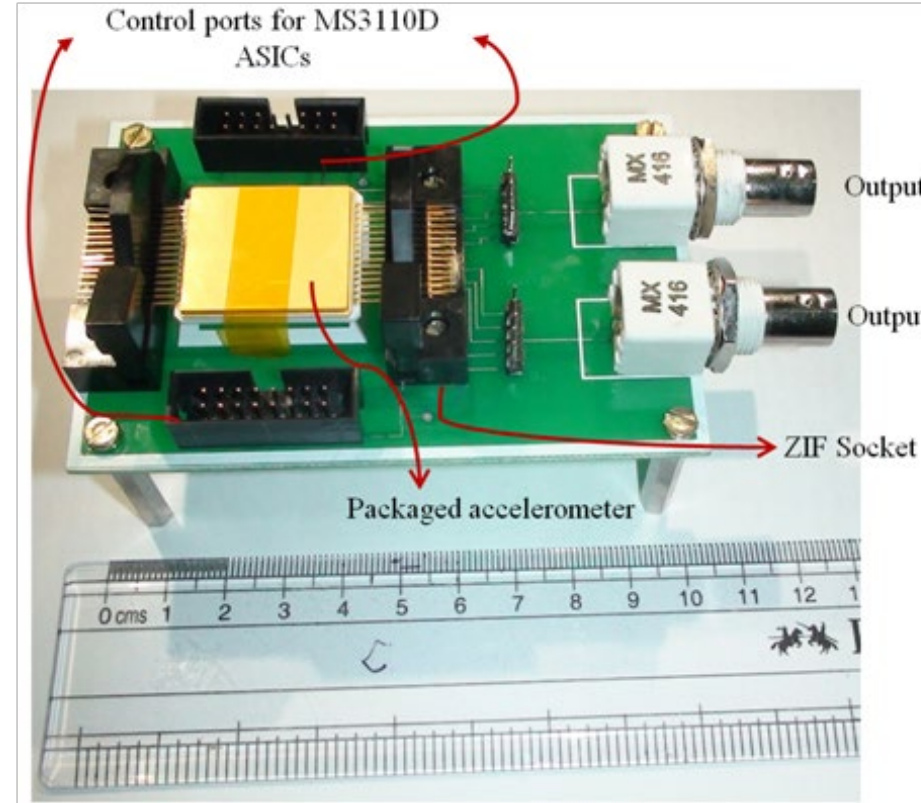
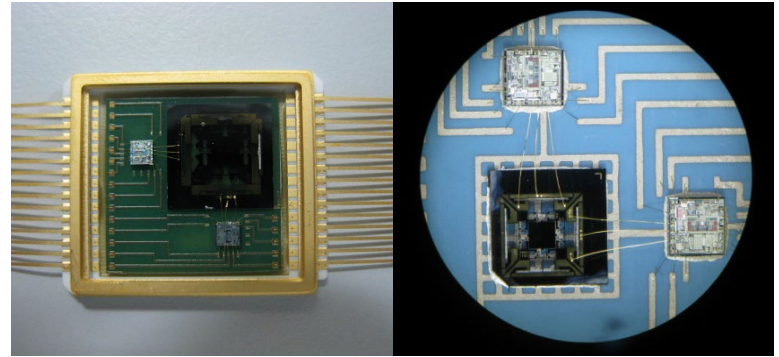
PCT/US2004/023211 dated 20th July, 2004 (H.C. Hermann and N. Mankame, G.K. Ananthasuresh)



# Improved micromachined accelerometers

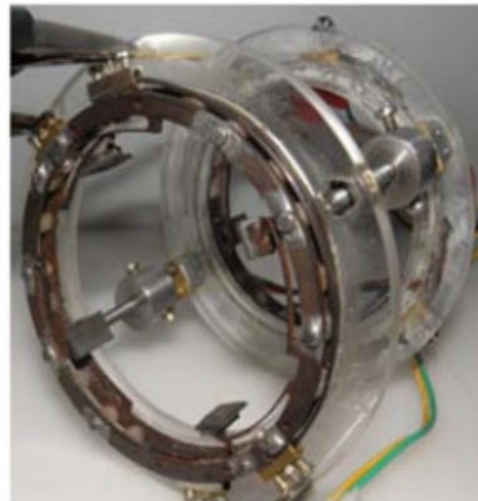
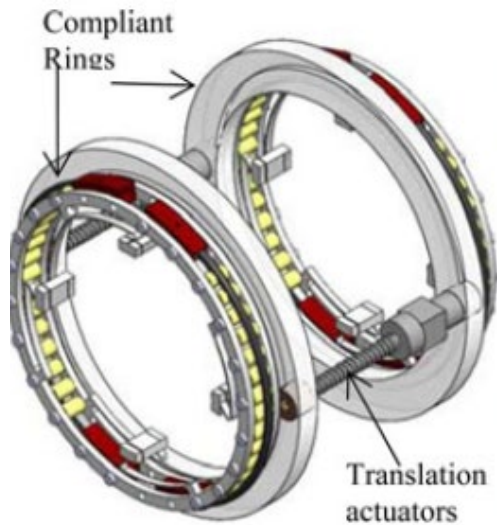
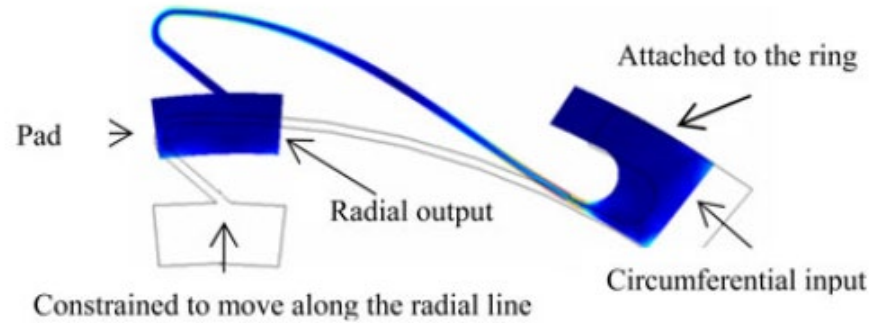


With Sambuddha Khan



# External pipe-crawler

Inchworm concept  
A compact ring-actuator



Step 1



Step 2



Step 3



Step 4



Step 5

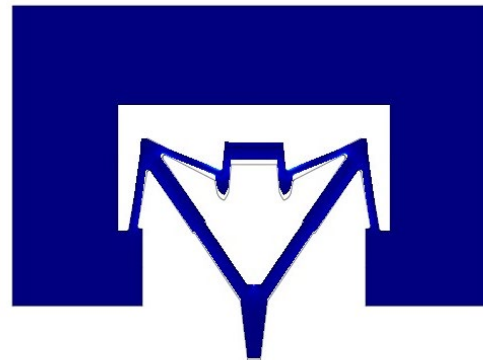
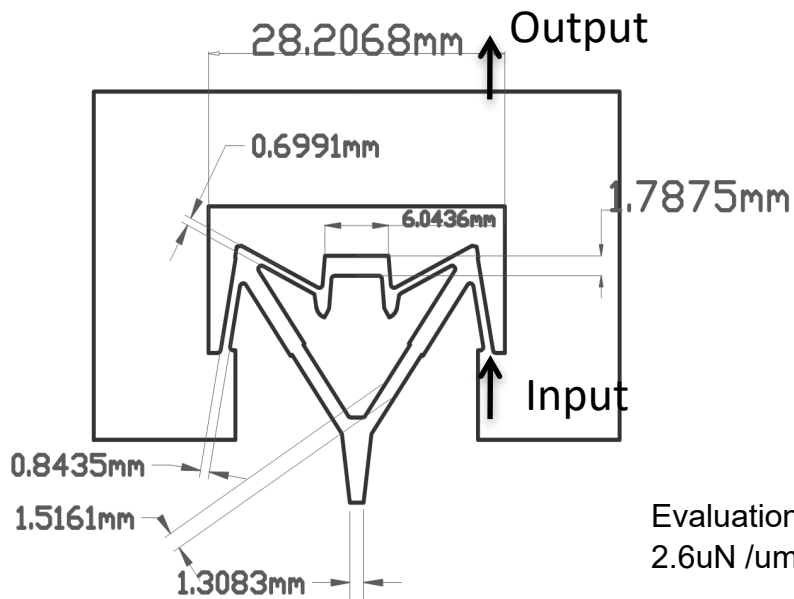
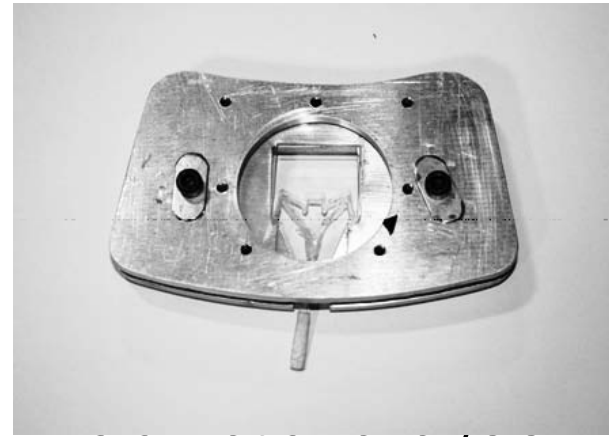


Next cycle

IEEE TRANSACTIONS ON ROBOTICS, VOL. 29, NO. 1, FEBRUARY 2013

With  
Puneet  
Singh et al.

# Micron: a force sensor



Evaluation: effective stiffness of 26.N/m or 2.6uN /um

~2  $\mu$ N resolution  
~ 1 mN range

Baichapur,  
Bhargav, Gulati,  
Maheswari, and  
Ananthasureh,  
2012

# Micron

Hello.

M2 Lab. Mech. Dept. IISc.



# These are good wheels!



Flattening of tire under loading

Non-pneumatic tire from IISc.

ME project report, G. Bhargav, 2008.



Flattening of tire under loading in *tweel*, an experimental design made by michelin and inspiration for this project

Tweel from Michelin.



Working of prototype

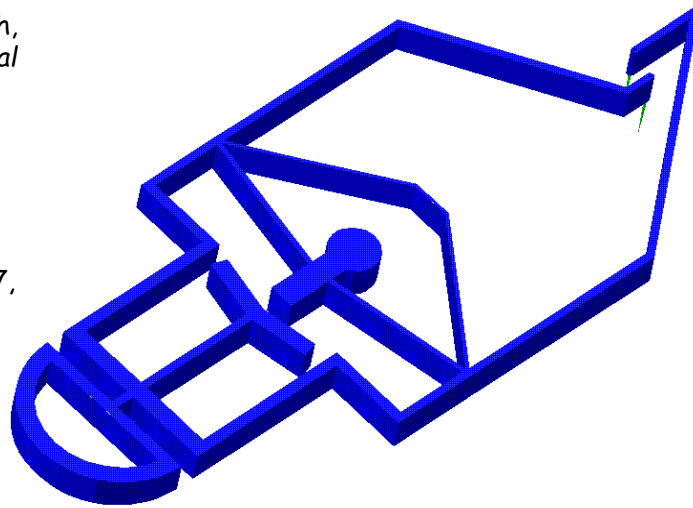
# Almost any type of motion

- Displacement amplification
- Curved paths
- Non-smooth paths
- Frequency tuning
- Constant-force
- Actuator characteristic modification
- Etc.
- Ease of manufacture – reduced or no assembly
- Ability to withstand overloads – “I bend but I break not.”
- Less or no friction and wear
- No backlash problem – more precision
- Aesthetics made easier
- Economy of material and less cost for better performance
- **Any type of motion**

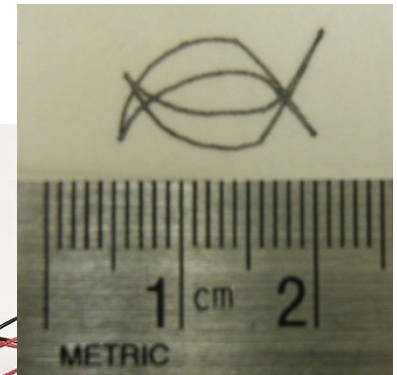
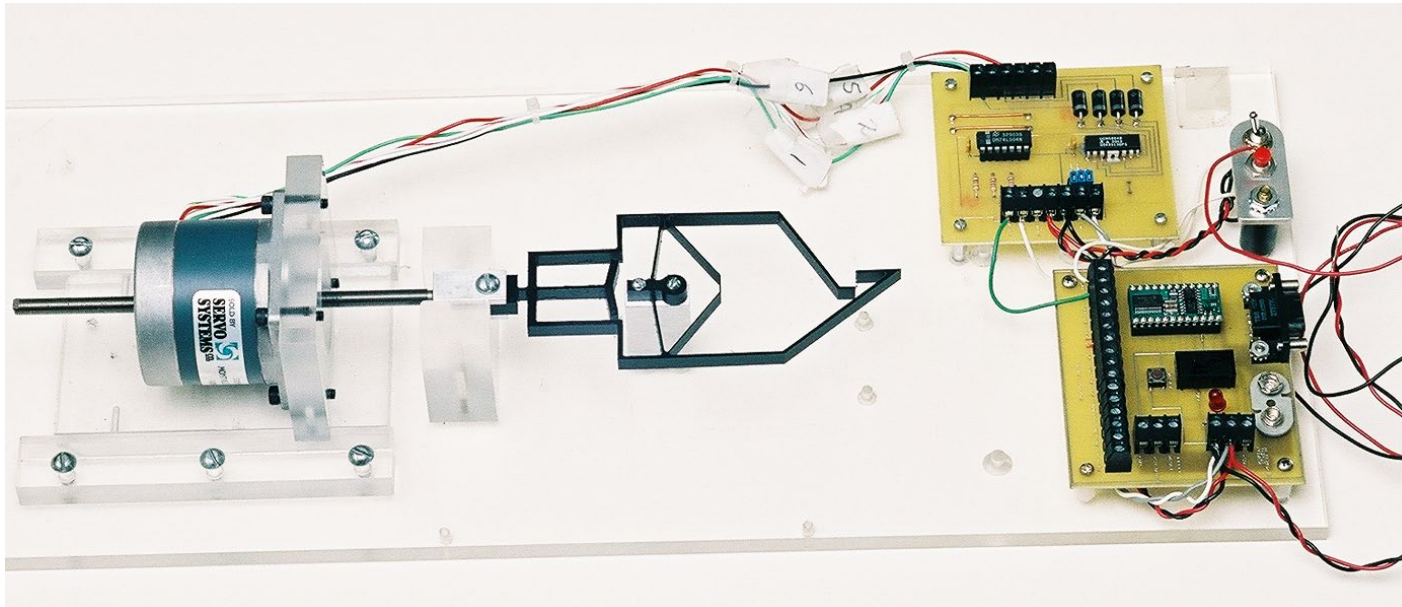
# Non-smooth curved paths

Mankame, N. and Ananthasuresh, G.K., *Journal of Mechanical Design*. 126(4), 2004, pp. 667-672.

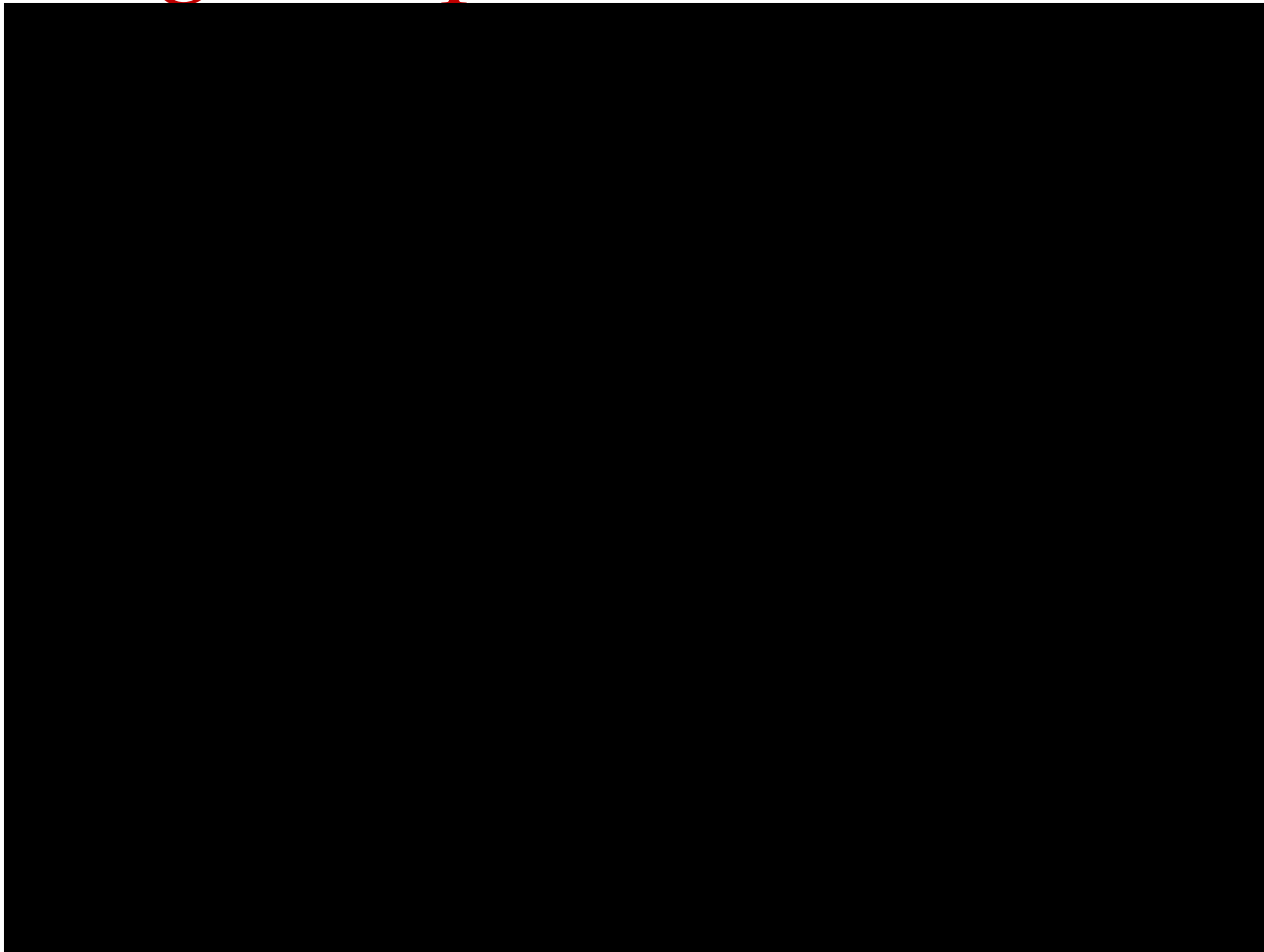
Mankame, N. D. and Ananthasuresh, G. K., *International Journal of Numerical Methods in Engineering*, 69 (12), 2007, pp. 2564-2605.



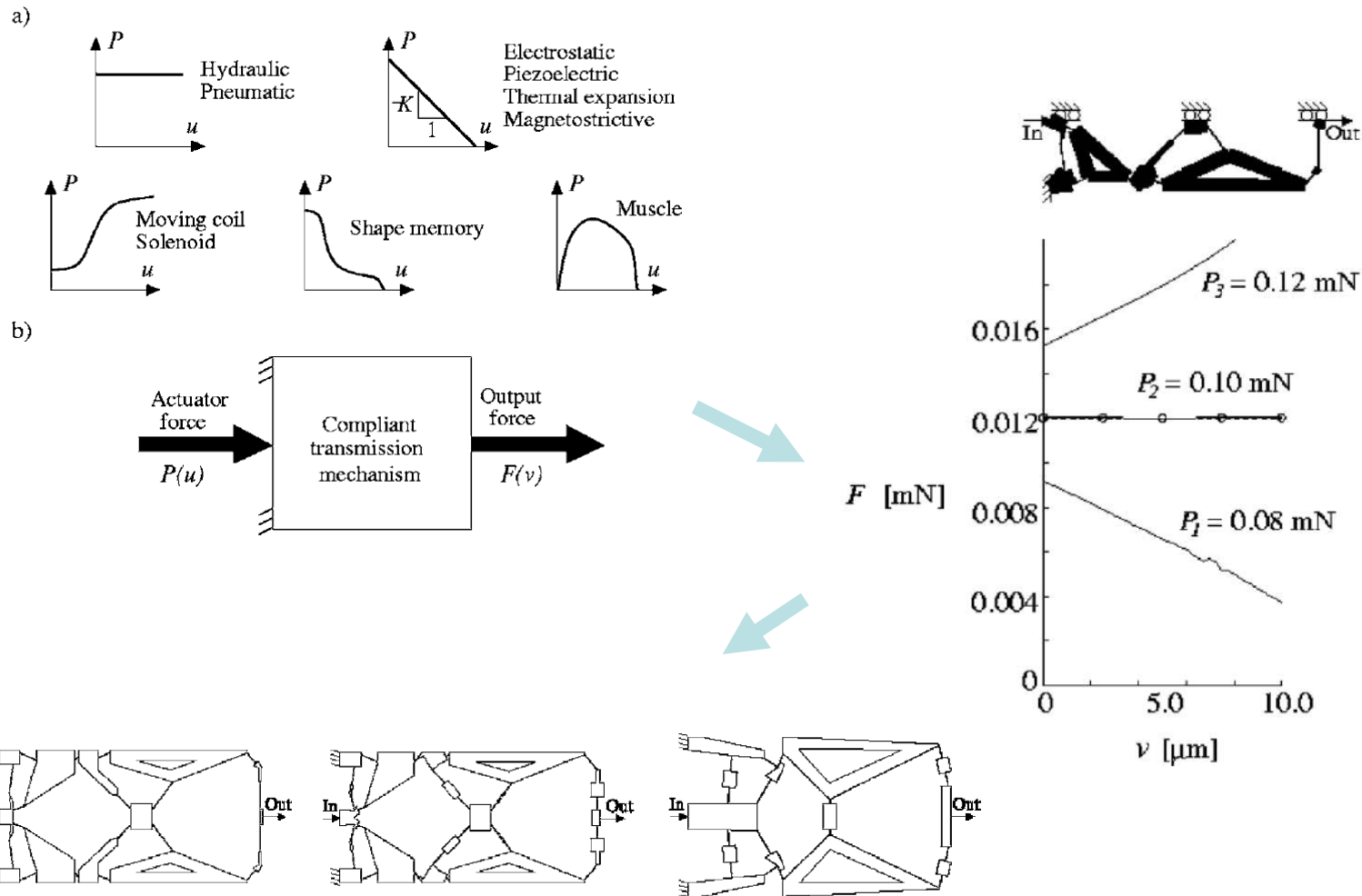
Movie



# Non-smooth motion with smooth input using compliant mechanisms



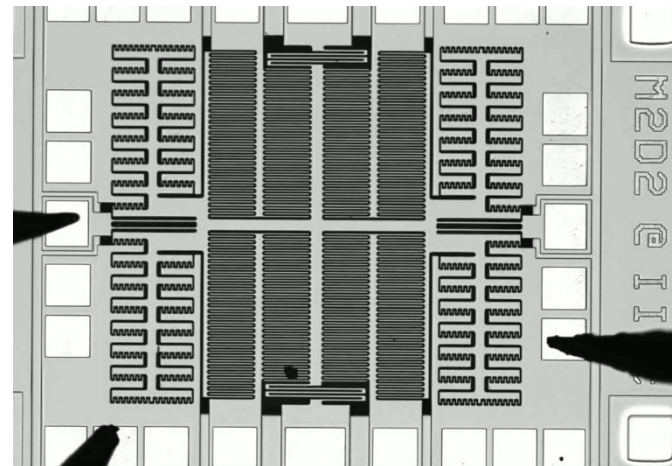
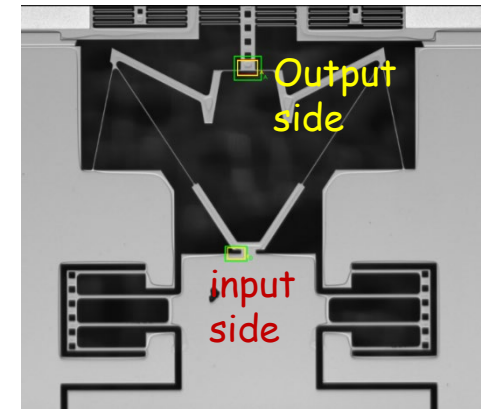
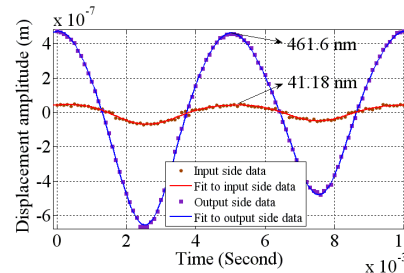
# Actuator characteristic modification



Pedersen, C. B. W., Fleck, N. A. and Ananthasuresh, G. K., *Journal of Mechanical Design*, 128(5), 2006, pp. 1101-1112.

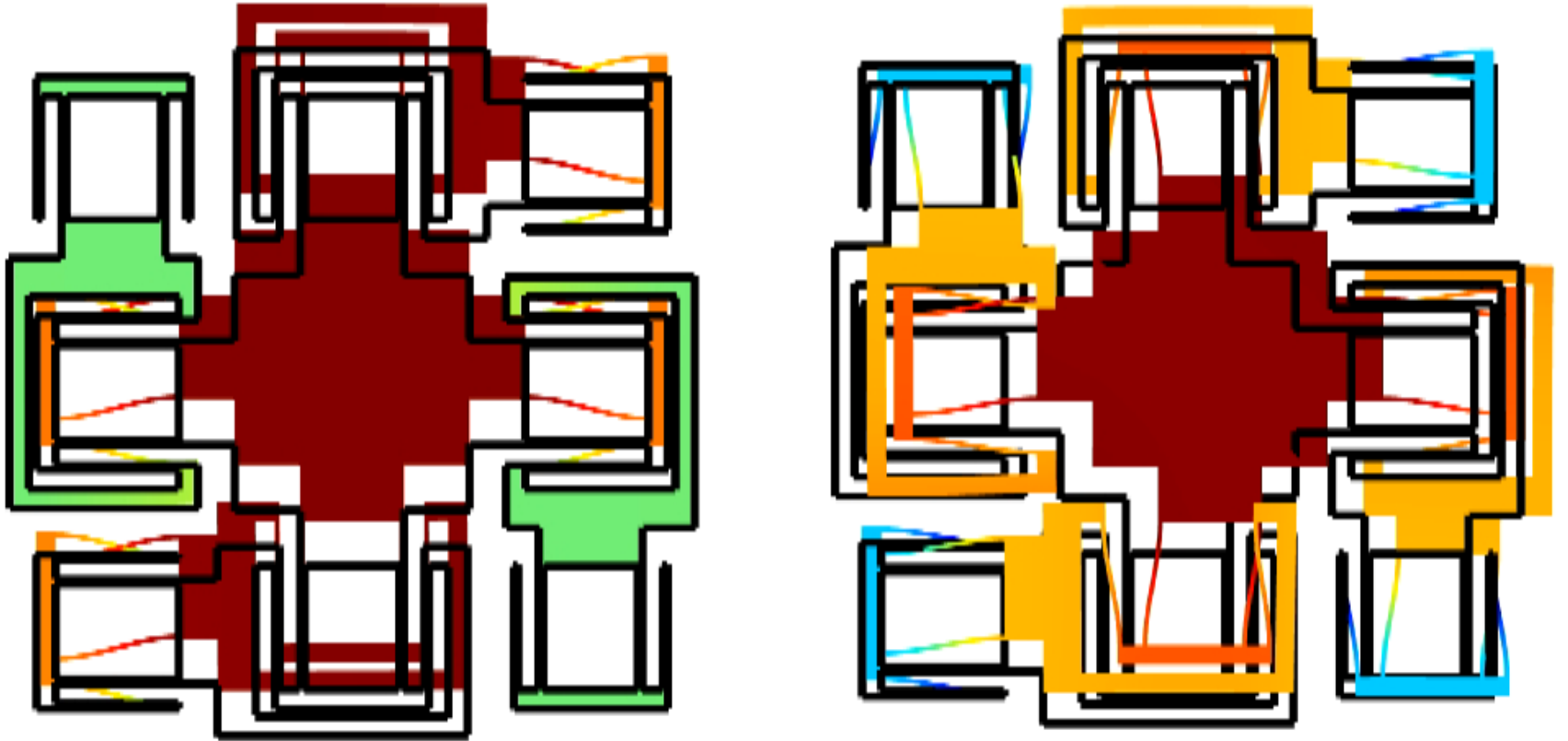
# Micromechanical signal processors

- Band-pass filters
- Switches and relays
- Amplifiers
- Frequency-translators
- Clocks



With  
Sambuddha  
Khan and  
Nirmit Dave

# Separating x-y signals



# Is compliant design difficult?

- Some people think so. It may be because...
  - you need to deal with elastic deformation.
  - you need to deal with **elastic pairs** and **elastic segments** as opposed to discrete rigid bodies.
- It is in fact easy once we pay attention to...
  - what benefits we can achieve with deformation
  - deformation mechanics in addition to kinematics



## Models for analysis

- Empirical modeling of elastic pairs
- Elastica analysis of beam segments
- Pseudo Rigid-Body (PRB) modeling
- **Finite element analysis**
- Spring-Lever (SL) and Spring-Mass-Lever (SML) modeling
- **Compliant ellipsoid modeling**
- **Non-dimensional maps**

## Synthesis methods

- Kinetostatic synthesis using PRB modeling
- **Topology and shape optimization**
- Selection and re-design
- Instant-centre method
- **Design using building blocks**
- **Pragmatic design with non-dimensional maps**
- Intuitive design using a kit